

MECCANICA DEGLI AZIONAMENTI

Presentazione 11: Oleodinamica

TRASMISSIONE MECCANICA



TRASMISSIONE OLEODINAMICA

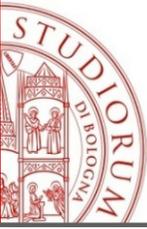


Vantaggi:

- potenza/ingombro $\uparrow\uparrow$, potenza/peso $\uparrow\uparrow$
- capacità di regolazione e controllo $\uparrow\uparrow$

Limiti:

- rendimenti inferiori
- apparati complessi e costosi per la sincronizzazione rigorosa dei movimenti
- precisione dei movimenti limitata a causa della comprimibilità del fluido e della variazione della viscosità con la temperatura



Richiami Meccanica dei Fluidi

$$p + \rho gh + \frac{1}{2} \rho w^2 = \text{costante}$$

$$Re = \frac{w \cdot d_h}{\nu} \quad Re_{cr} \approx 2300 \quad \text{Se } Re < Re_{cr} \text{ il moto è laminare; se } Re > Re_{cr} \text{ il moto è turbolento.}$$

$$\Delta p = \frac{1}{2} \lambda \rho w^2 \frac{L}{d}$$

$$\Delta p = \frac{1}{2} \beta \rho w^2$$

a) $\lambda_{li} = \frac{64}{Re}$ nel caso del regime laminare isotermico;

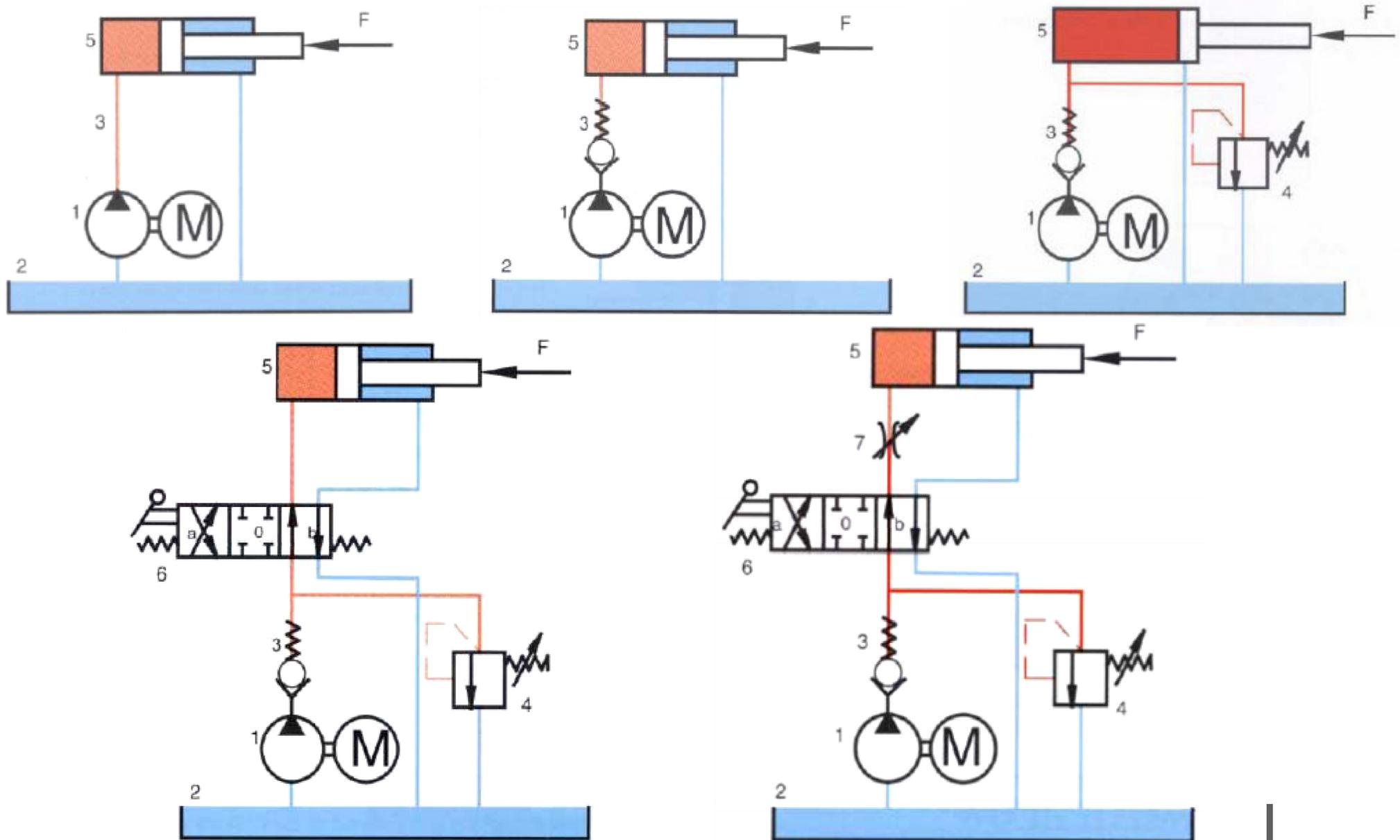
b) $\lambda_{la} = \frac{75}{Re}$ nel caso del regime laminare adiabatico;

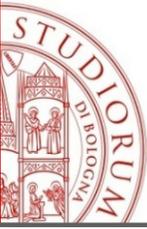
c) $\lambda_{turb} = \frac{0.316}{\sqrt[4]{Re}}$ nel caso del regime turbolento.

$$P = Q \Delta p$$

Configurazione	β
Gomito 90°	0.9
Gomito 45°	0.5
Giunzione a T fluente a 90°	1.8
Giunzione a T fluente a 180°	0.9
Valvola a saracinesca	0.2
Valvola a sfera	10
Contrazione brusca	1
Espansione brusca	0.5

Circuito elementare





Pompe e Motori Idraulici

Pompe volumetriche

$$n_p \cdot \frac{\text{giri}}{\text{min}}, \quad \frac{n_p}{60} \cdot \frac{\text{giri}}{\text{s}}$$

$$Q_{th} = V_p \frac{n_p}{60} \quad \left[\frac{\text{m}^3}{\text{s}} \right]$$

$$Q = \eta_{vp} Q_{th} = \eta_{vp} V_p \frac{n_p}{60} \quad \left[\frac{\text{m}^3}{\text{s}} \right]$$

$$P_{id} = Q \Delta p_p = \eta_{vp} Q_{th} \Delta p_p = \eta_{vp} V_p \frac{n_p}{60} \Delta p_p \quad [\text{W}]$$

$$P_p = \frac{P_{id}}{\eta_{vp} \eta_{mp}} = \frac{Q_{th} \Delta p_p}{\eta_{mp}} = \frac{V_p n_p \Delta p_p}{60 \eta_{mp}} \quad [\text{W}]$$

$$\eta_{mp} = \eta_{mp}(n_p, \Delta p_p)$$

$$C_p = \frac{P_p}{\omega_p} = \frac{V_p n_p \Delta p_p}{60 \eta_{mp}} \cdot \frac{60}{2\pi n_p} = \frac{V_p \Delta p_p}{2\pi \eta_{mp}} \quad [\text{Nm}]$$

Motori idraulici rotativi

$$Q = \frac{V_m n_{thm}}{60} \Rightarrow n_{thm} = \frac{60Q}{V_m} \quad \left[\frac{\text{giri}}{\text{min}} \right]$$

$$n_m = \eta_{vm} n_{thm} = \eta_{vm} \frac{60Q}{V_m} \quad \left[\frac{\text{giri}}{\text{min}} \right]$$

$$P_{id} = Q \Delta p_m \quad [\text{W}]$$

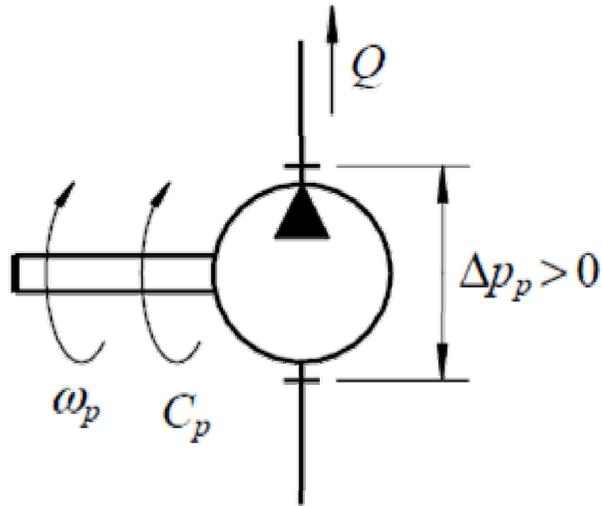
$$P_m = \eta_{vm} \eta_{mm} P_{id} = \eta_{vm} \eta_{mm} Q \Delta p_m = \eta_{mm} \frac{V_m n_m \Delta p_m}{60} \quad [\text{W}]$$

$$C_m = \frac{P_m}{\omega_m} = \eta_{mm} \frac{V_m n_m \Delta p_m}{60} \cdot \frac{60}{2\pi n_m} = \eta_{mm} \frac{V_m \Delta p_m}{2\pi} \quad [\text{Nm}]$$

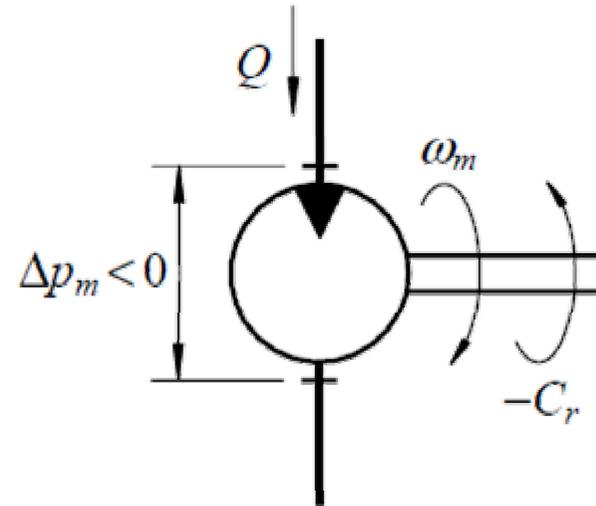
$$C_m = C_r \Rightarrow \Delta p_m = \frac{2\pi C_r}{\eta_{mm} V_m} \quad [\text{Pa}]$$

Pompe e Motori Idraulici

Pompa oleodinamica:



Motore oleodinamico:



*Unidirezionale
a portata fissa*



*Unidi-rezionale
a portata variabile*



*Bidirezionale
a portata fissa*



*Unidirezionale
a portata fissa*

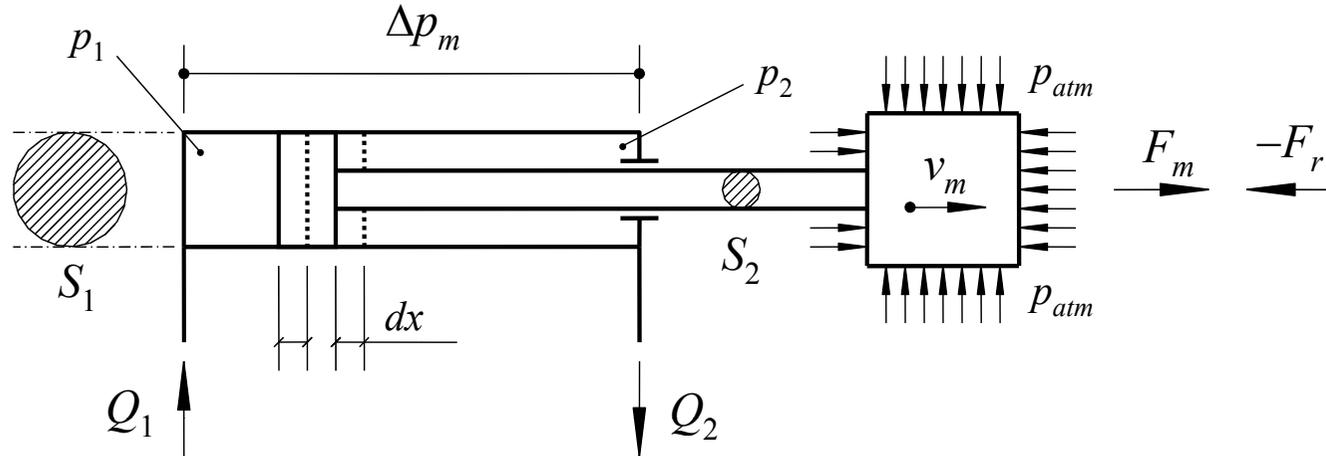


*Unidi-rezionale
a portata variabile*



*Bidirezionale
a portata fissa*

Motori Idraulici Lineari (Cilindri)



$$\eta_{vm} dV_1 = S_1 dx \Rightarrow \eta_{vm} Q_1 = S_1 v_m \Rightarrow v_m = \eta_{vm} \frac{Q_1}{S_1} \quad \left[\frac{\text{m}}{\text{s}} \right]$$

$$Q_1 = \text{cost.} \Rightarrow v_m = \text{cost.} = \frac{\Delta x}{\Delta t} \Rightarrow \Delta t = \frac{\Delta x}{v_m} = \frac{S_1}{\eta_{vm} Q_1} \Delta x$$

$$dV_2 = (S_1 - S_2) dx + (1 - \eta_{vm}) dV_1$$

$$Q_2 = (S_1 - S_2) v_m + (1 - \eta_{vm}) Q_1 = (S_1 - S_2) \eta_{vm} \frac{Q_1}{S_1} + (1 - \eta_{vm}) Q_1 =$$

$$= Q_1 \left(1 - \frac{S_2}{S_1} \eta_{vm} \right)$$

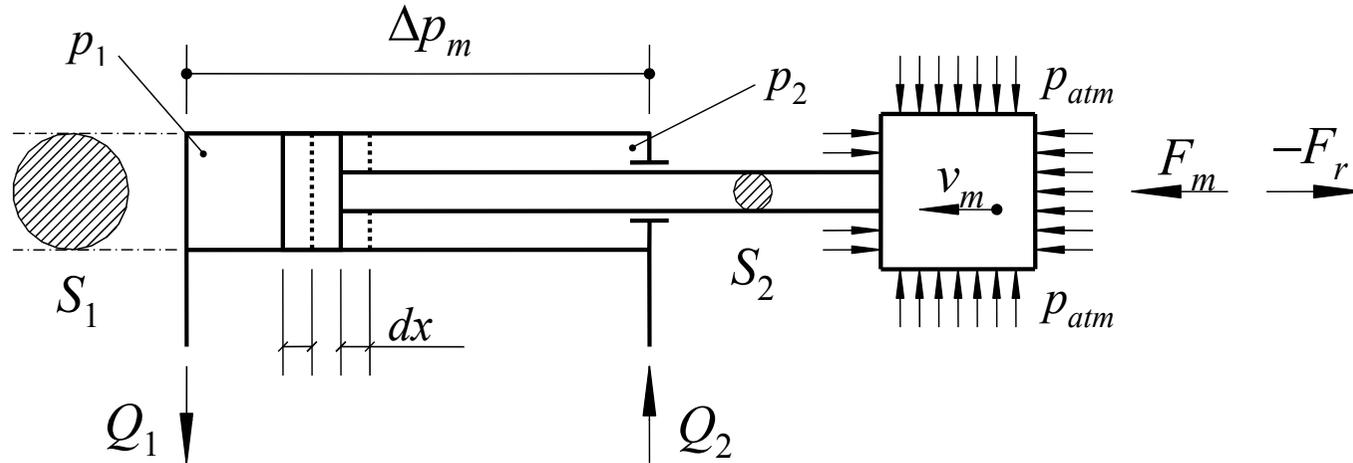
$$F_m = \eta_{mm} [p_1 S_1 - p_2 (S_1 - S_2)] =$$

$$= \eta_{mm} [(p_1 - p_2) S_1 + p_2 S_2] =$$

$$= \eta_{mm} [\Delta p_m S_1 + p_2 S_2] \quad [\text{N}]$$

$$F_m = F_r \Rightarrow \Delta p_m \approx \frac{F_r}{\eta_{mm} S_1} \quad [\text{Pa}]$$

Motori Idraulici Lineari (Cilindri)



Fase rientro stelo:

$$v_m = \eta_{vm} \frac{Q_2}{S_1 - S_2} \quad \left[\frac{\text{m}}{\text{s}} \right]$$

$$dV_1 = S_1 dx + (1 - \eta_{vm}) dV_2$$

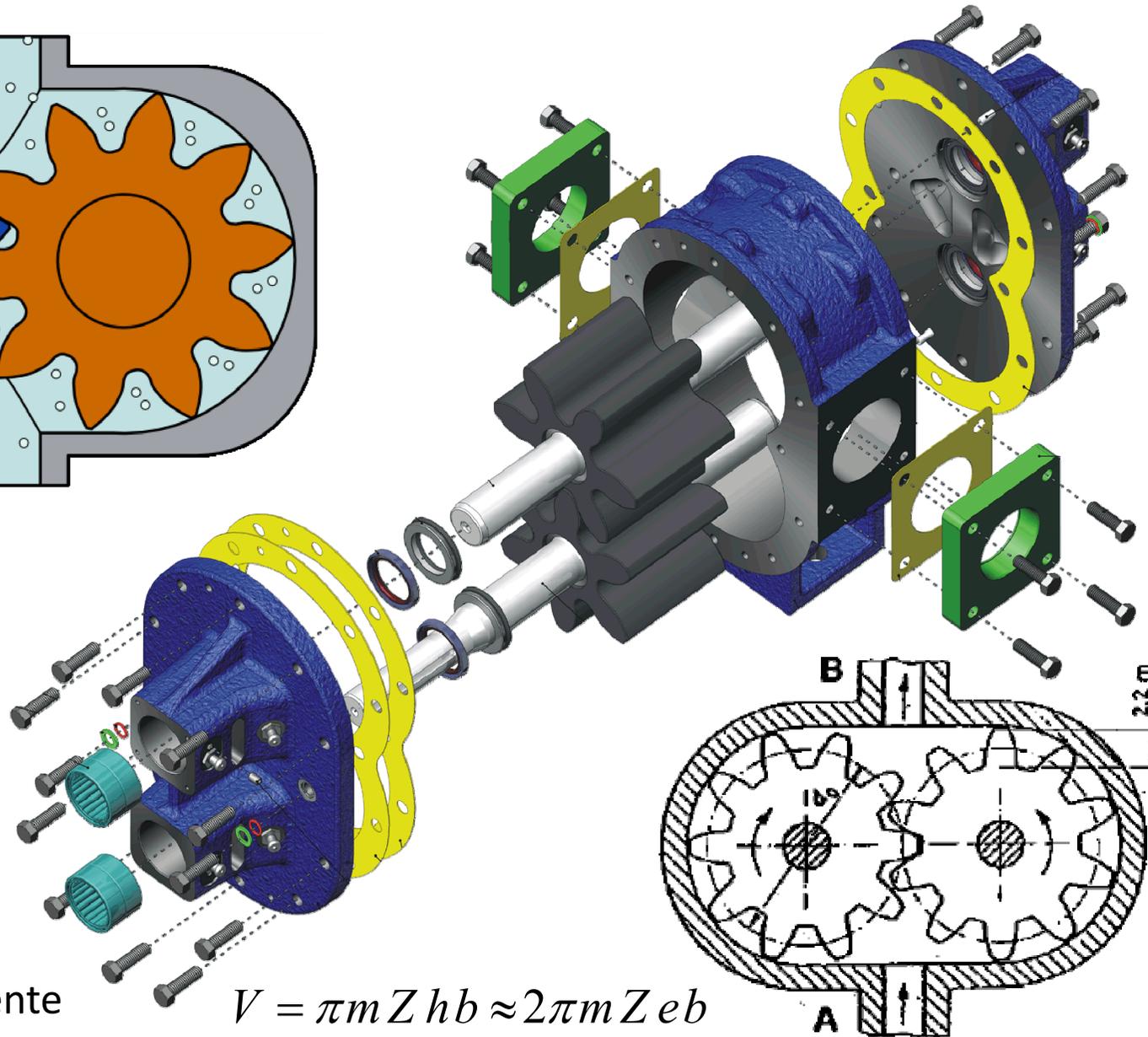
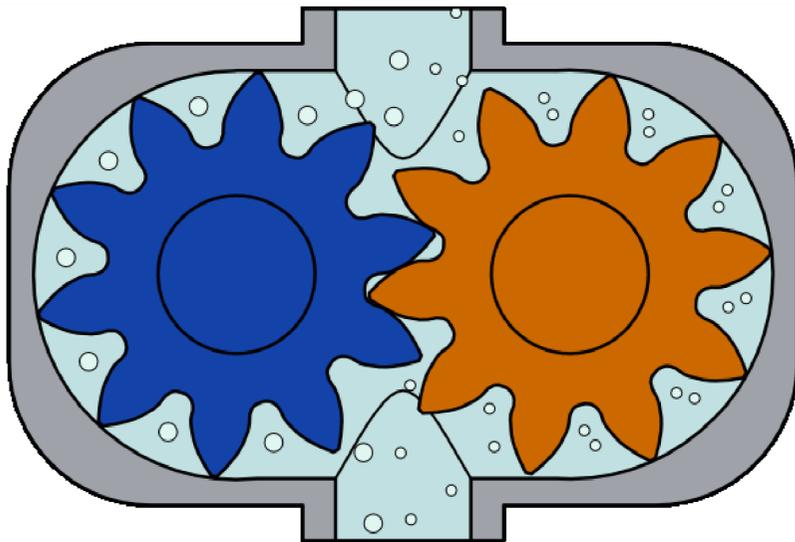
$$Q_1 = Q_2 \left(1 + \frac{S_2}{S_1 - S_2} \eta_{vm} \right)$$

$$F_m = \eta_{mm} [p_2 (S_1 - S_2) - p_1 S_1] \quad [\text{N}]$$

$$p_2 \approx 0, \quad F_m = F_r$$

$$\Rightarrow \Delta p_m \approx \frac{F_r}{\eta_{mm} (S_1 - S_2)} \quad [\text{Pa}]$$

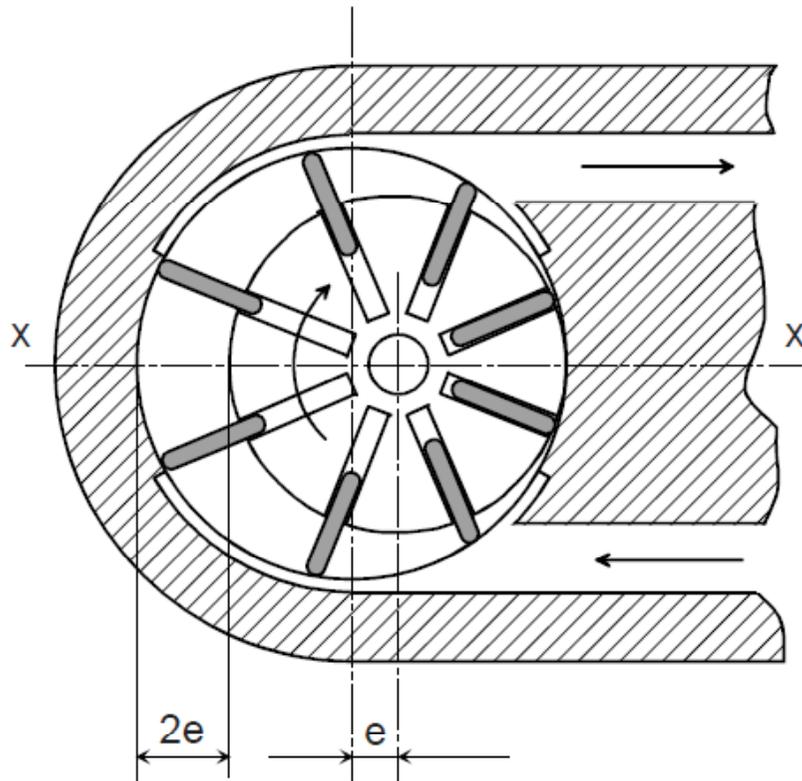
Pompa a ingranaggi



e = addendum dente

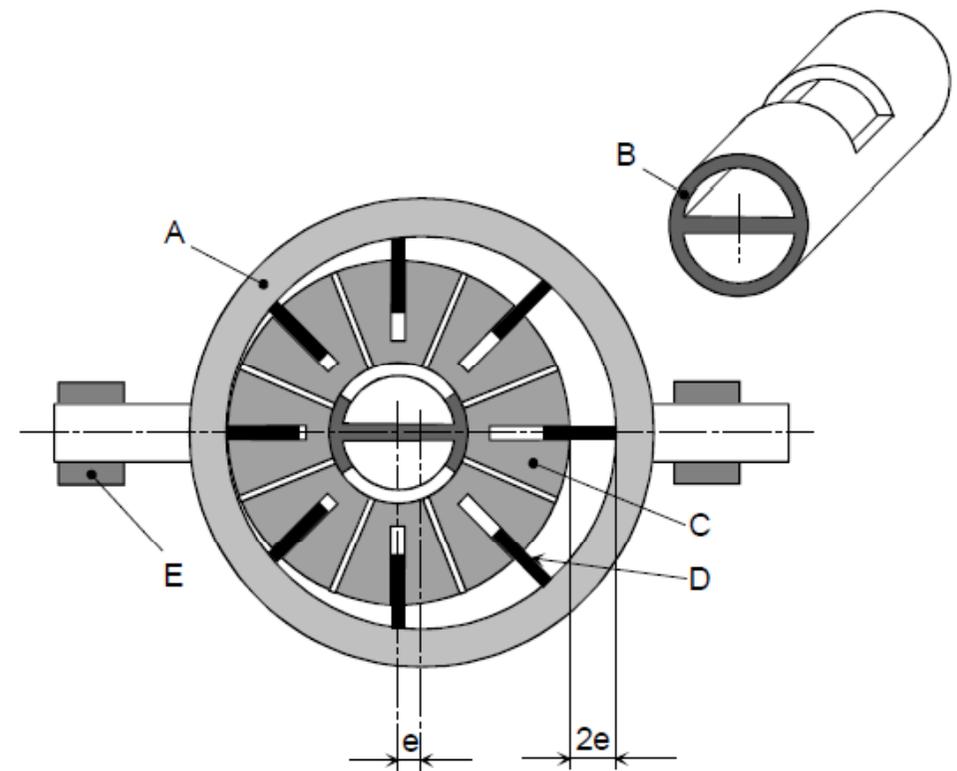
$$V = \pi m Z h b \approx 2\pi m Z e b$$

Pompa a palette

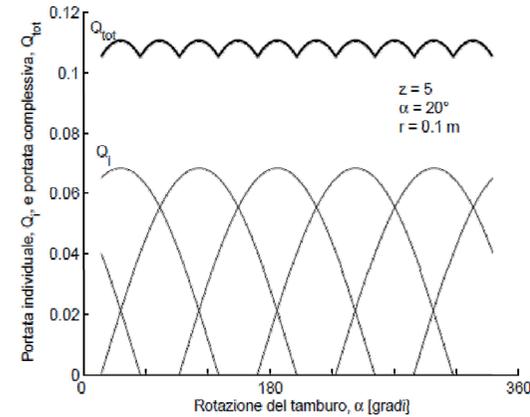
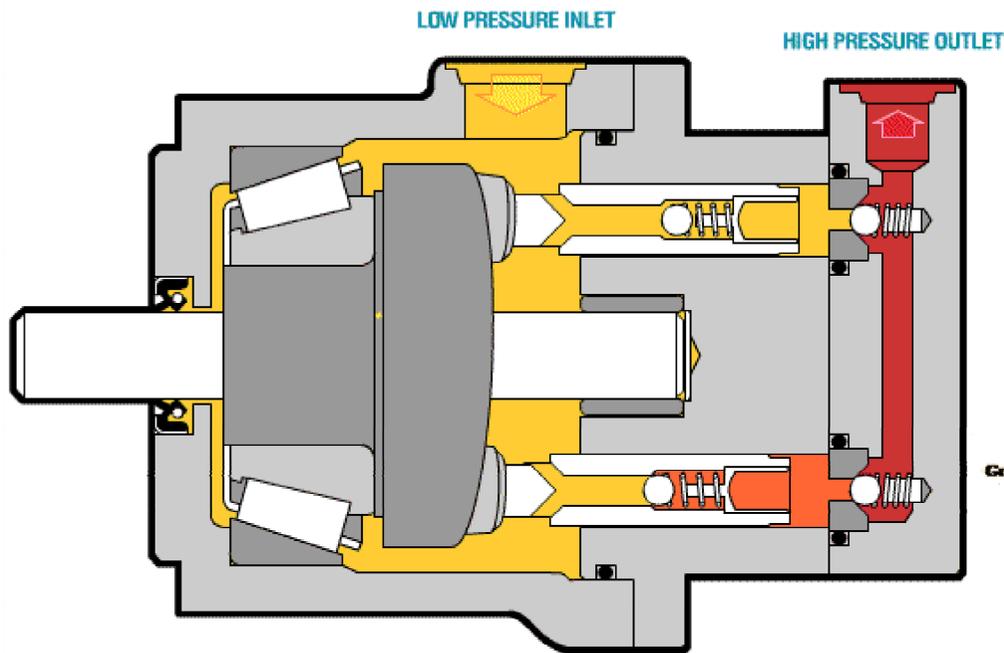


$$V_F = d_m \pi 2eb$$

$$d_m = d + e$$

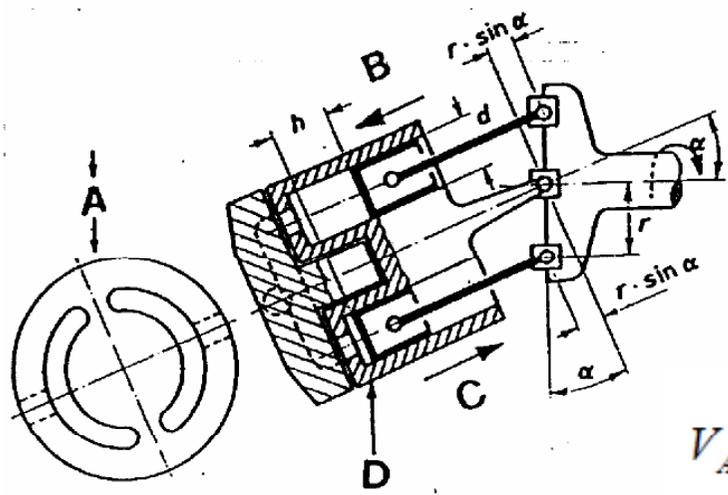
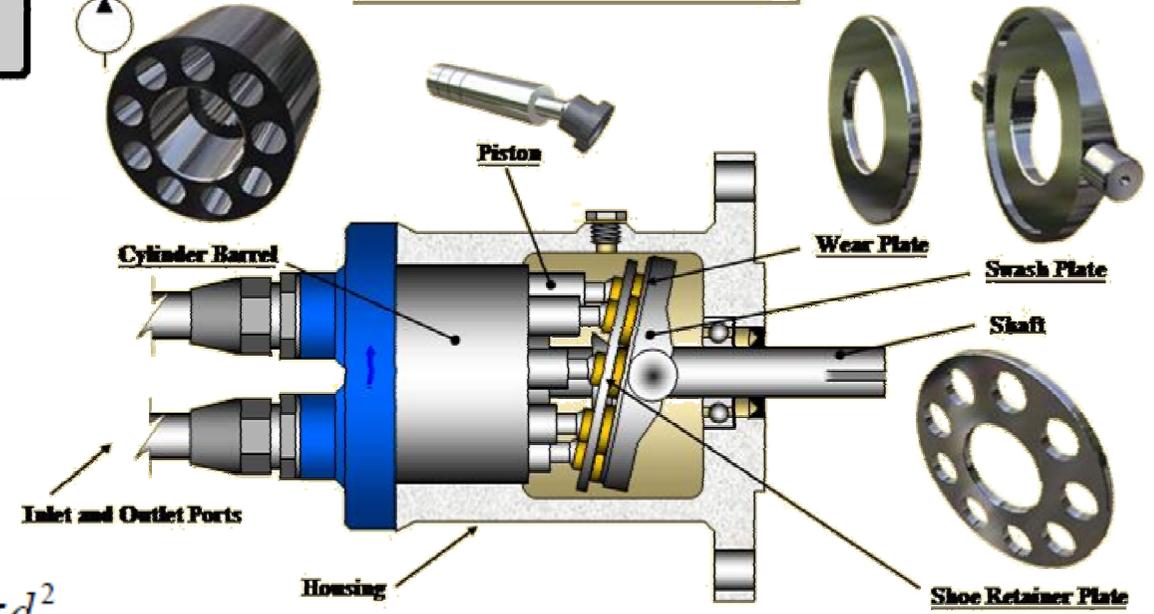


Pompa a pistoni assiali



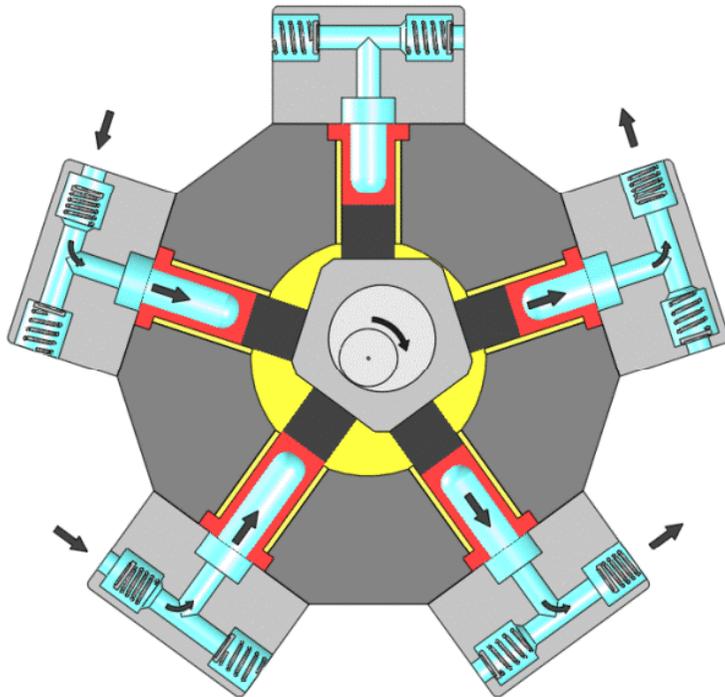
Axial Piston Pump

Graphic Symbol



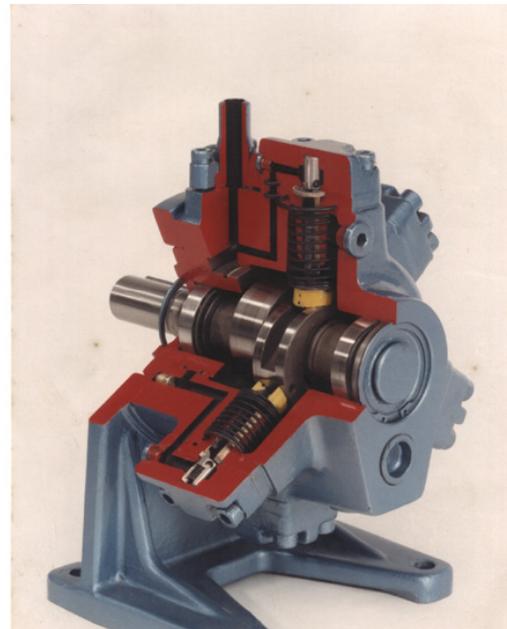
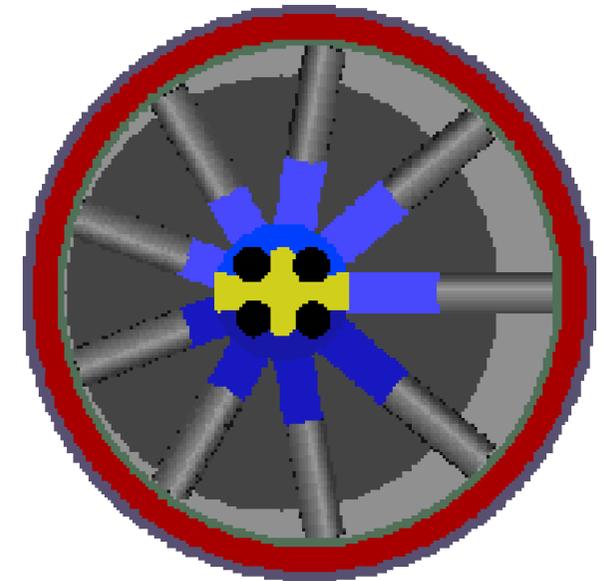
$$V_A = z \frac{\pi d^2}{4} 2r \sin \alpha$$

Pompa a pistoni radiali

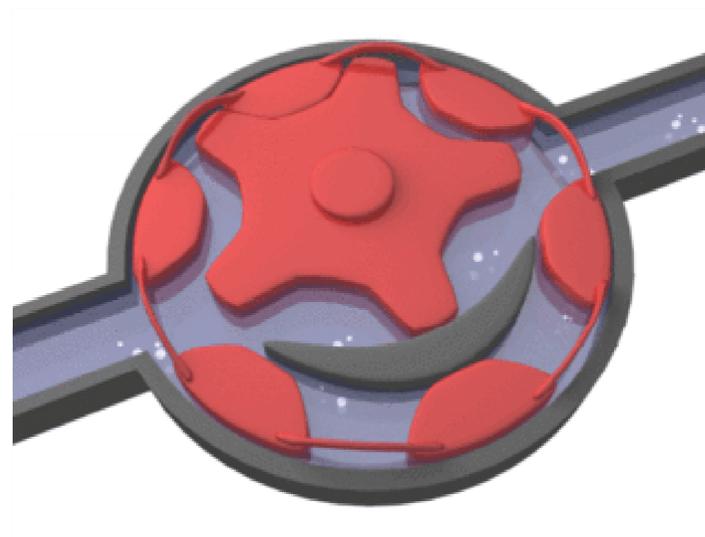
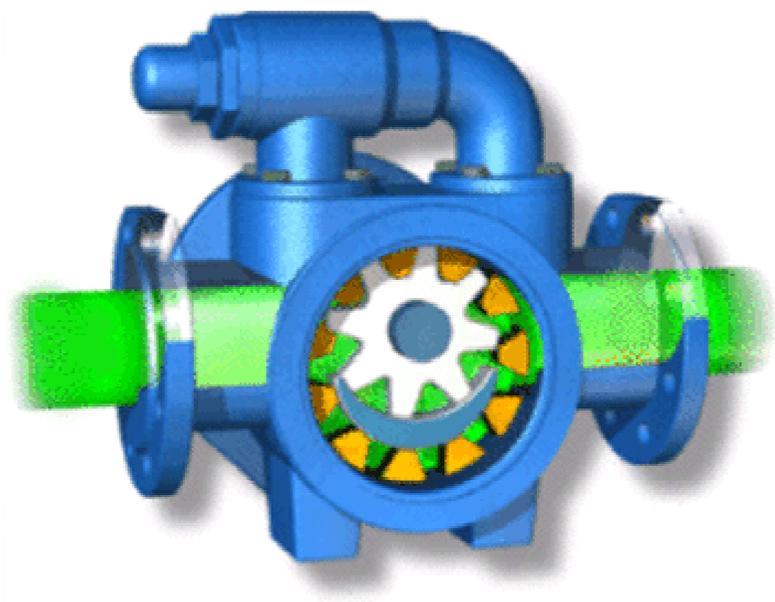


$$V = \frac{1}{2} \pi d^2 e Z$$

d = alesaggio cilindri

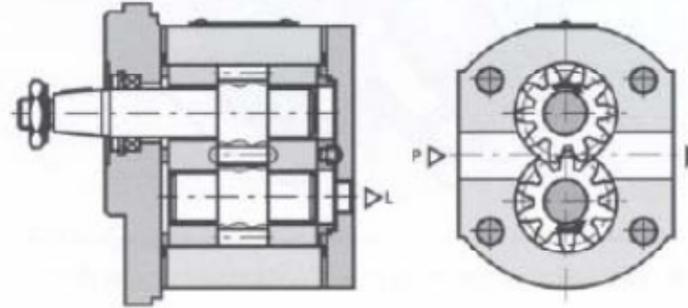


Pompa Orbitale (Gerotor)

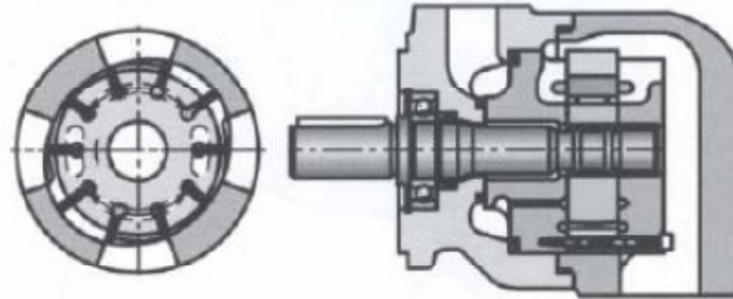


Motori Idraulici

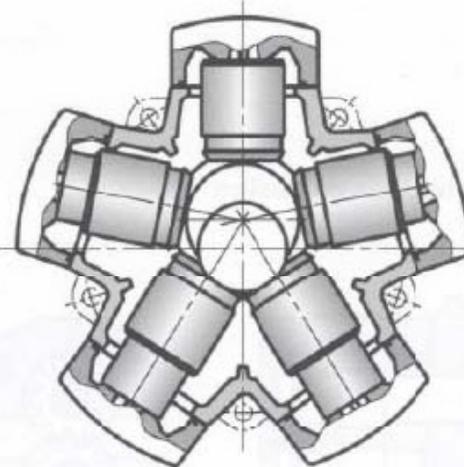
Motore ad ingranaggi



Motore a palette

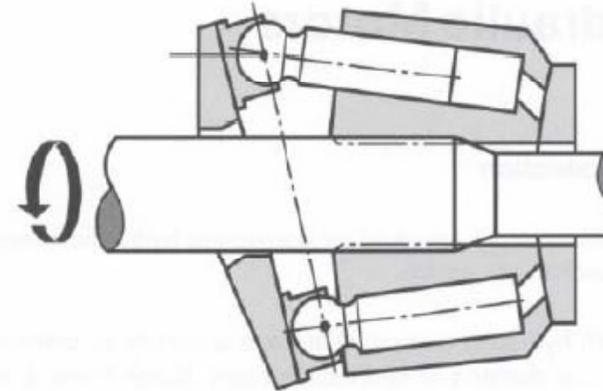


Motore a pistoni radiali con
eccentrico interno

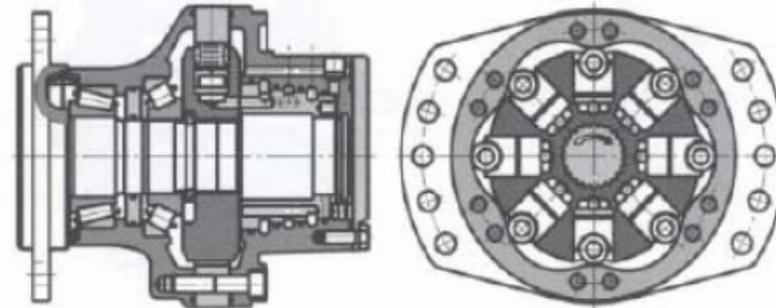


Motori Idraulici

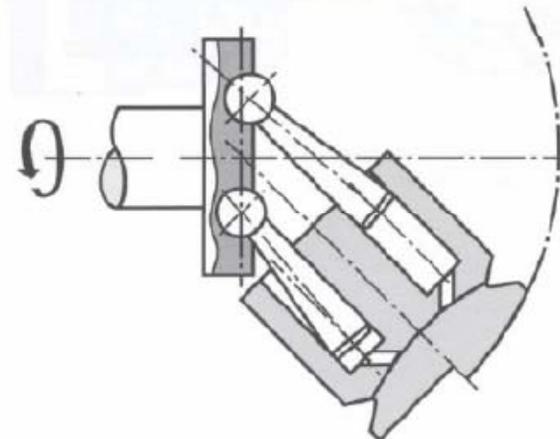
Motore a pistoni assiali con
piatto oscillante



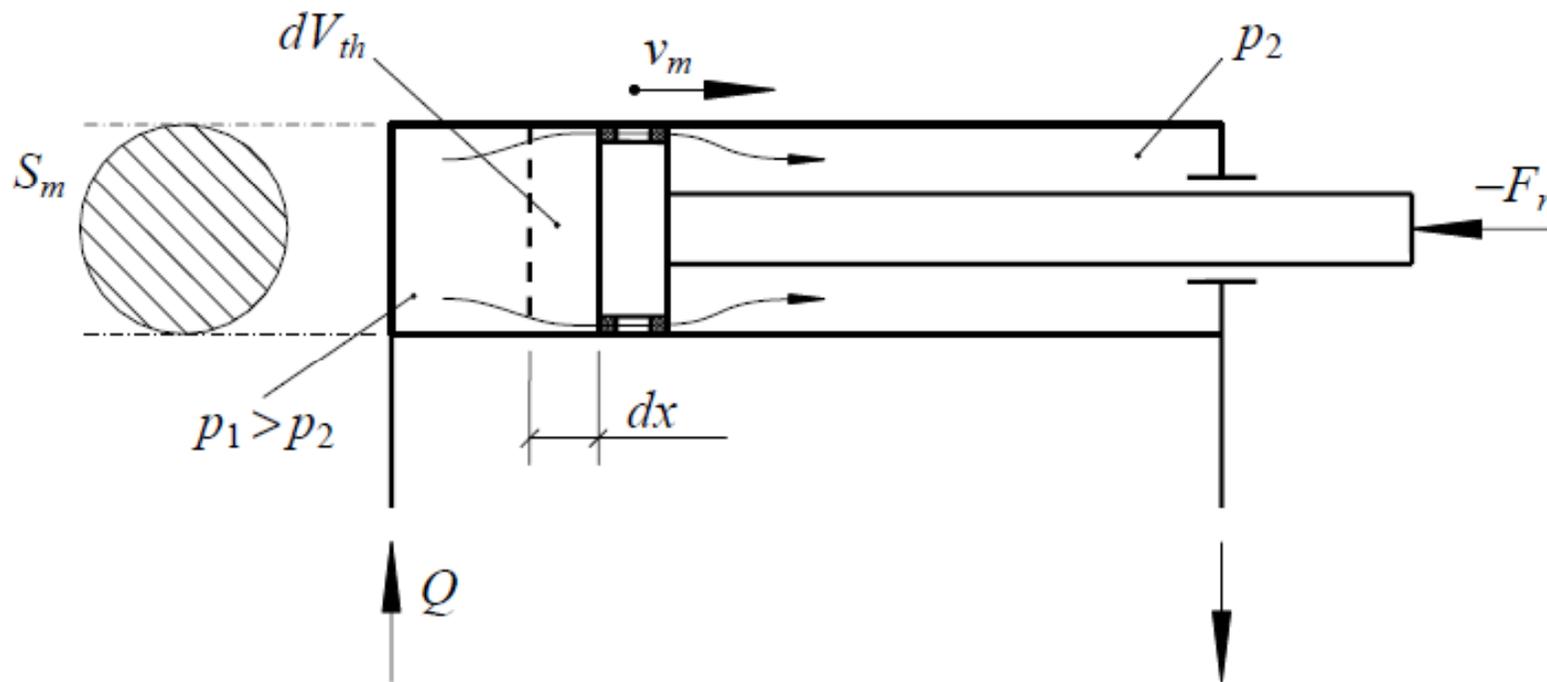
Motore a pistoni radiali con
canna esterna



Motore a pistoni assiali con
asse inclinato



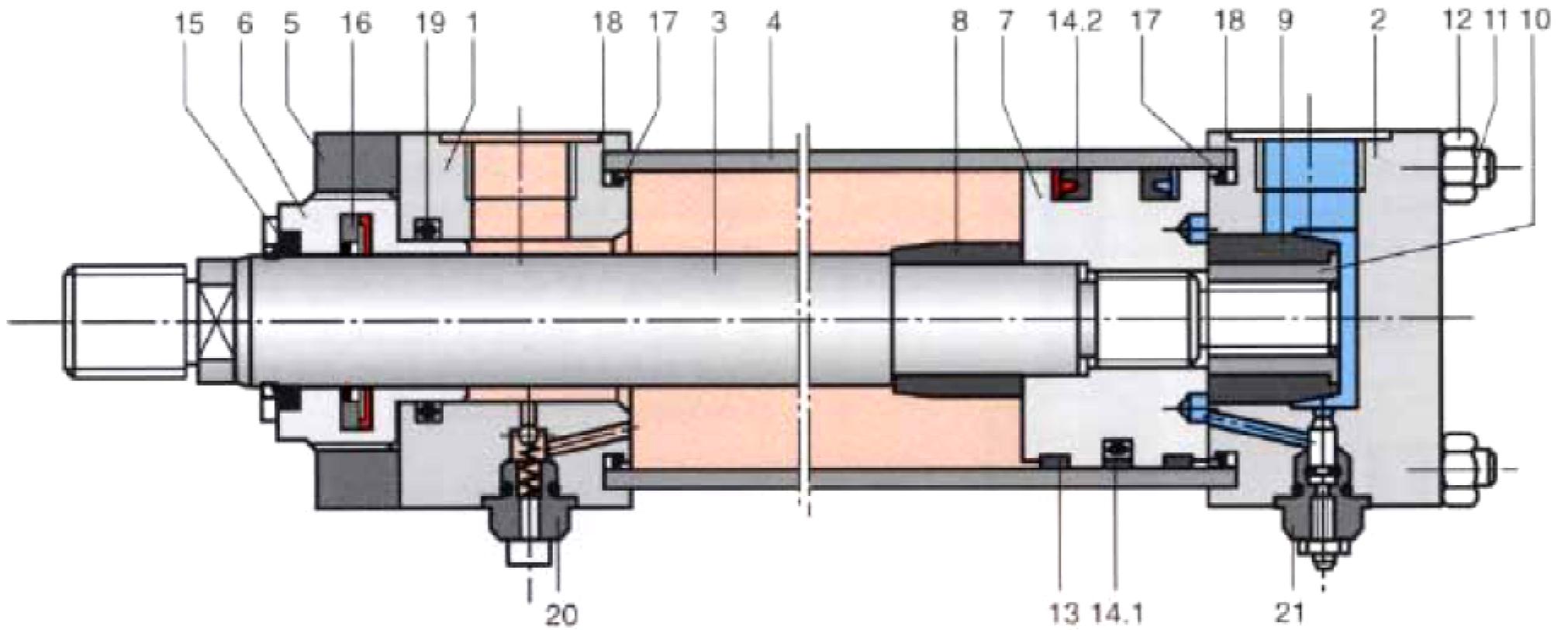
Cilindri Oleodinamici



$$dV_m = \eta_v dV_{th} = \eta_v Q dt, \quad \eta_v < 1$$

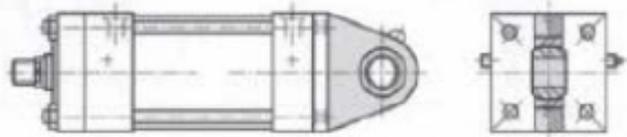
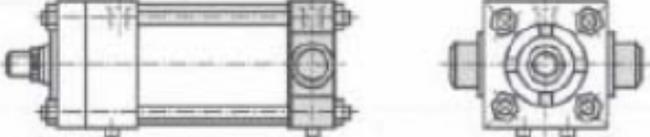
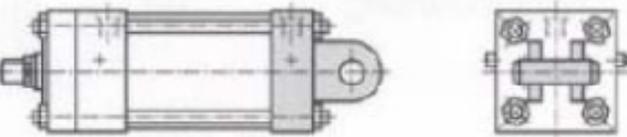
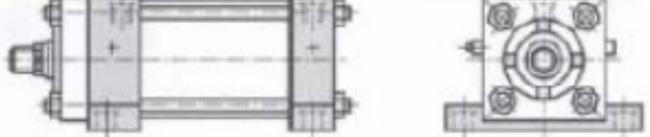
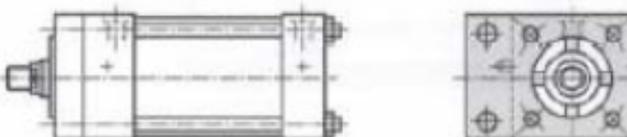
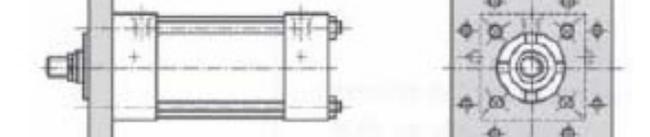
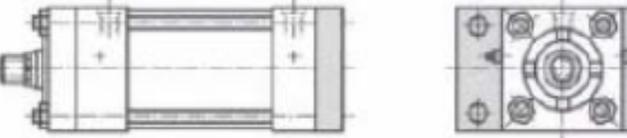
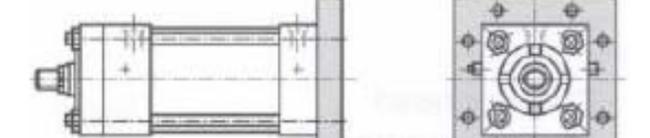
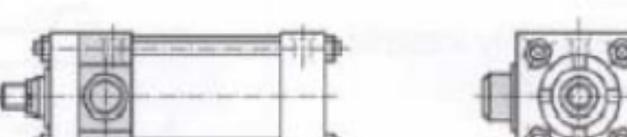
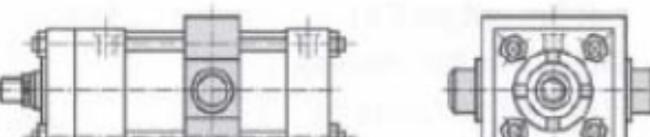
$$dV_m = S_m dx = S_m v_m dt \quad \Rightarrow \quad v_m = \eta_v \frac{Q}{S_m}$$

Cilindri Oleodinamici



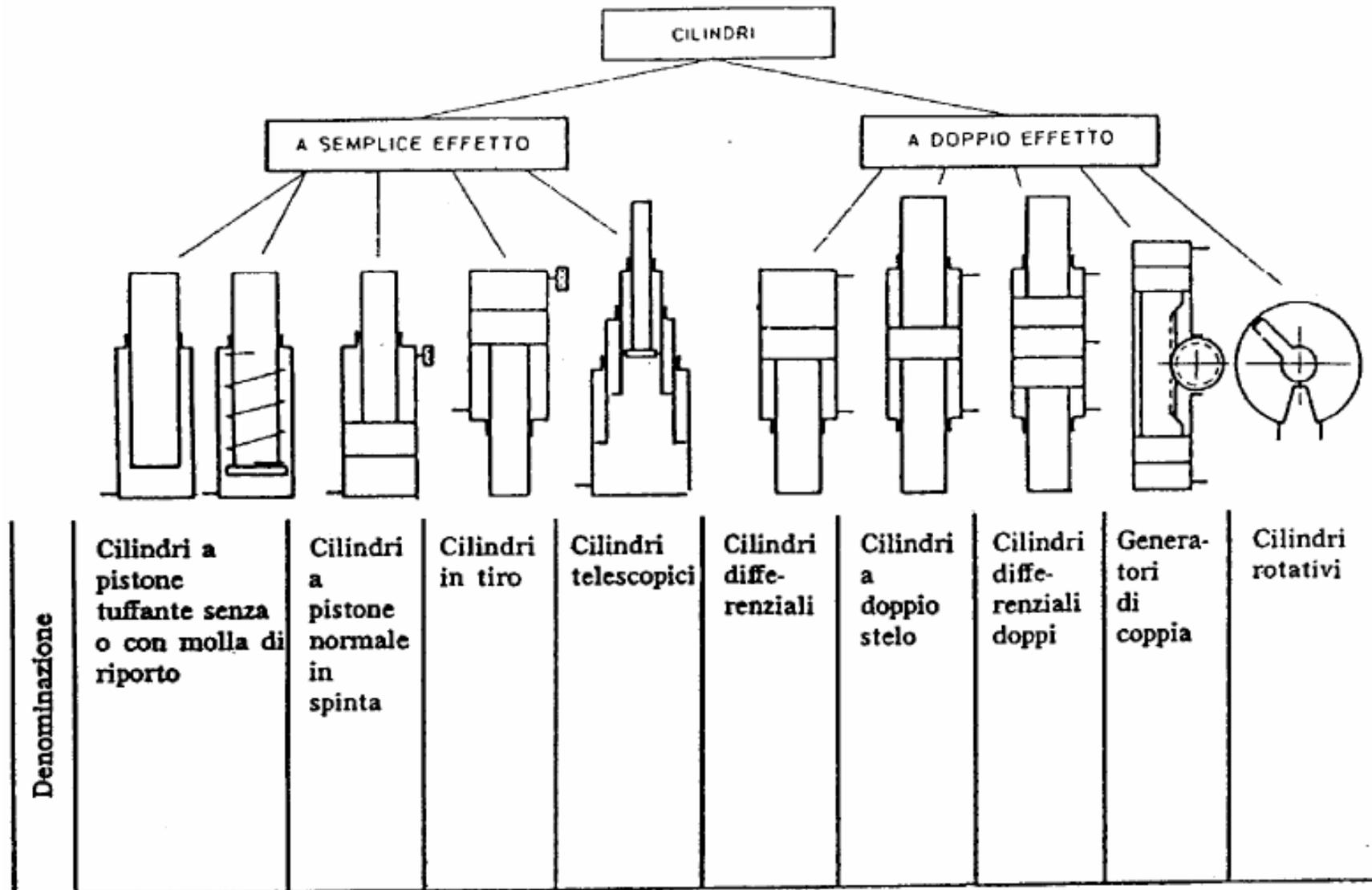
Cilindri Oleodinamici

Sistemi di fissaggio

<p>Staffa semplice alla base</p>		<p>Perni alla base</p>	
<p>Staffa doppia alla base</p>		<p>Montaggio con piedini</p>	
<p>Flangia rettangolare in testa</p>		<p>Flangia quadrata in testa</p>	
<p>Flangia rettangolare alla base</p>		<p>Flangia quadrata alla base</p>	
<p>Perni in testa</p>		<p>Perni al centro</p>	

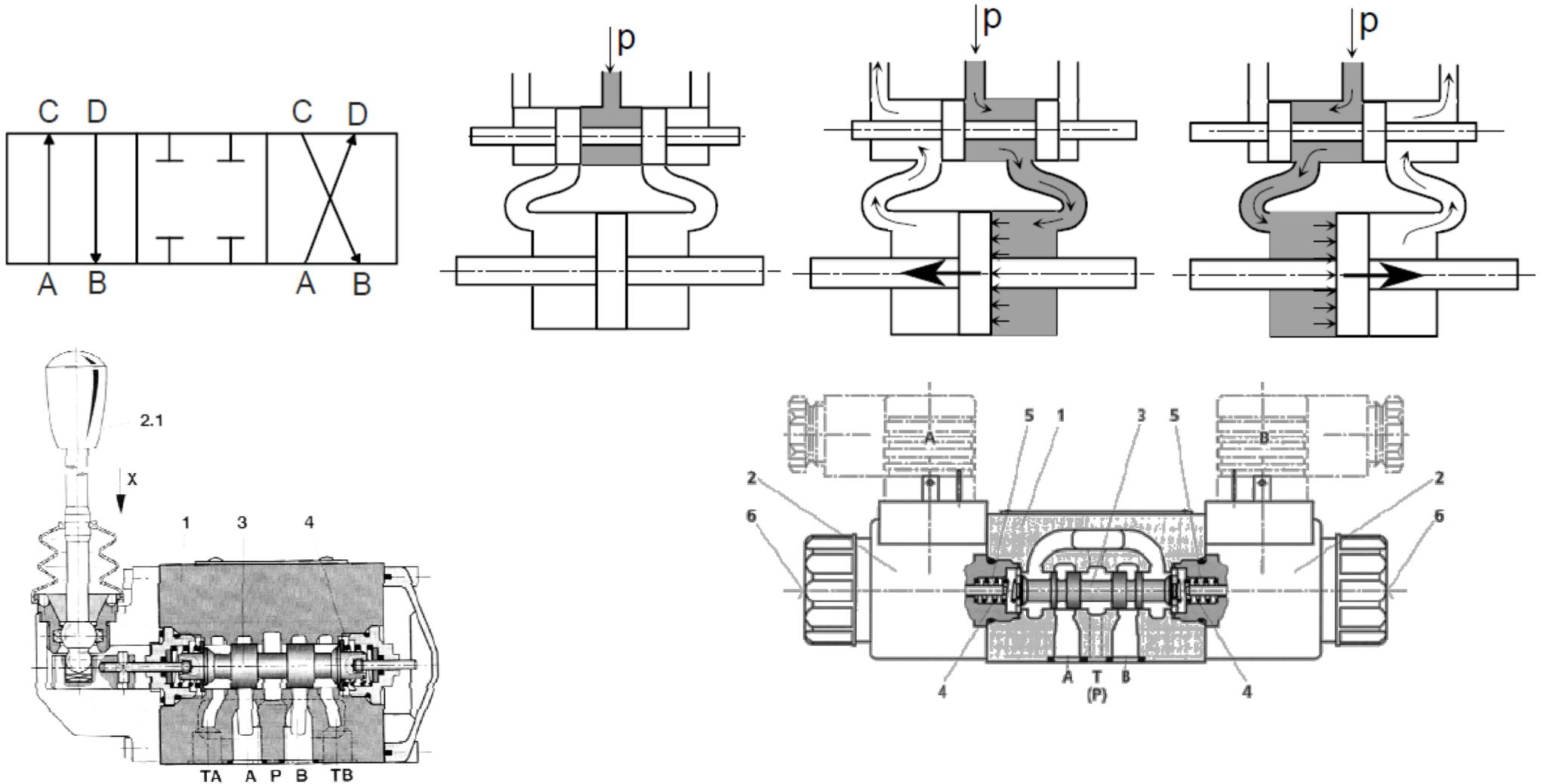
Cilindri Oleodinamici

Tipo di movimento



Altri componenti

Valvole di distribuzione

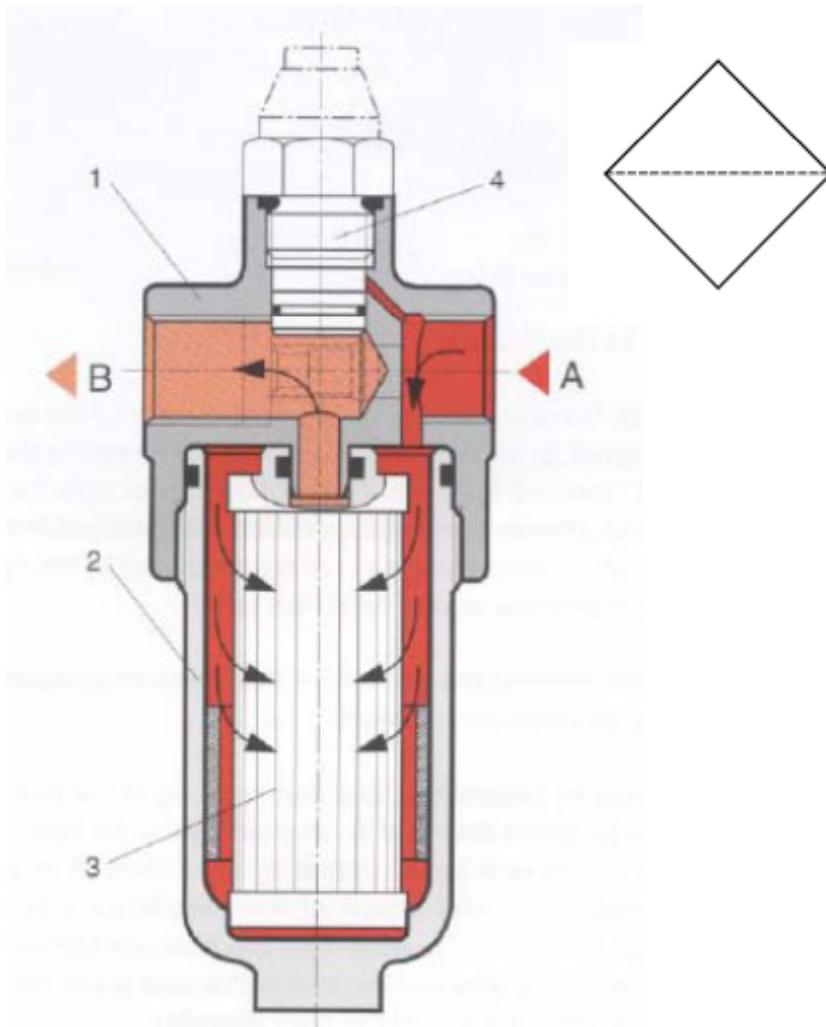


a) distributore 4/3 a comando manuale

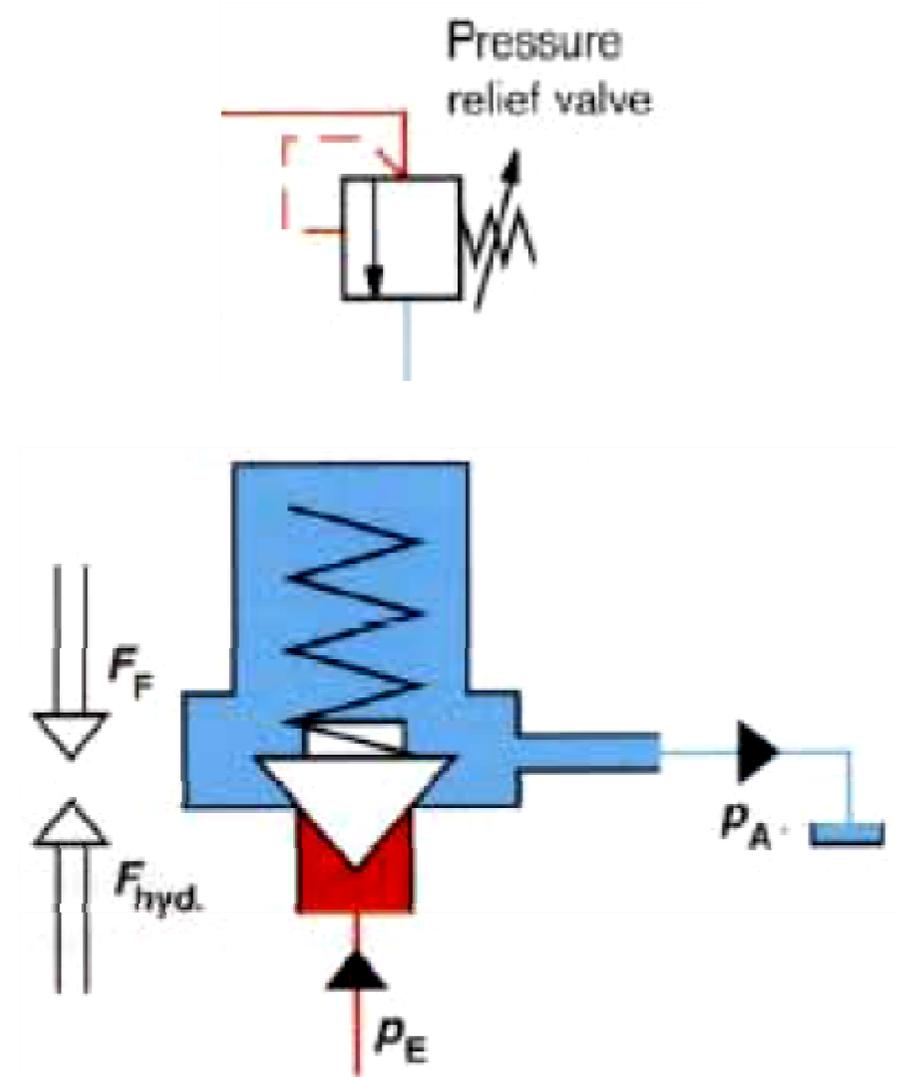
b) distributore 4/3 con solenoidi

Altri componenti

Filtri

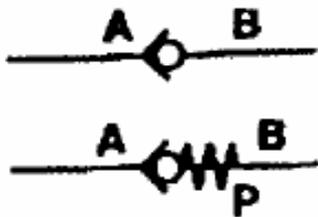
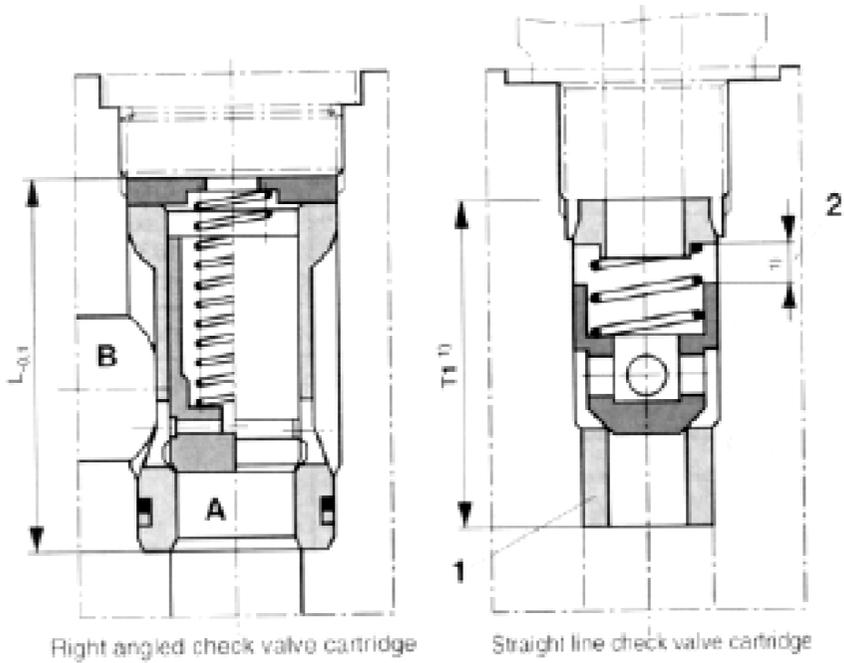


Limitatori di pressione

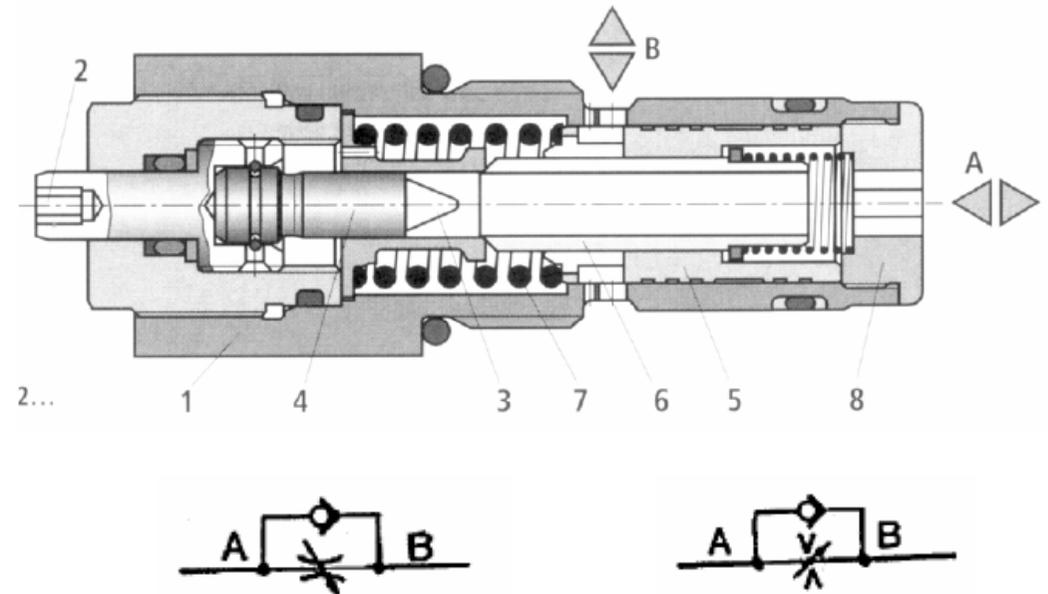


Altri componenti

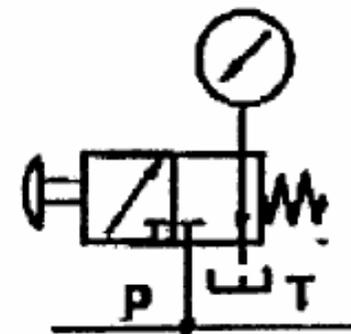
Valvole di non ritorno



Regolatori di portata



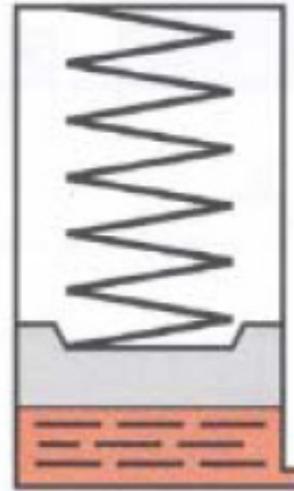
Manometri



Altri componenti

Accumulatori

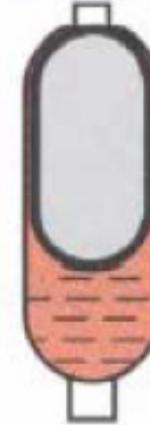
Elemento separatore tra gas e liquido



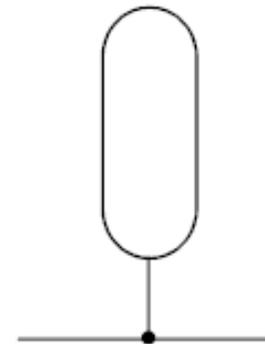
pistone



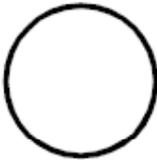
sacca



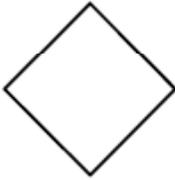
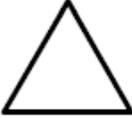
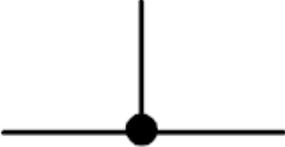
membrana



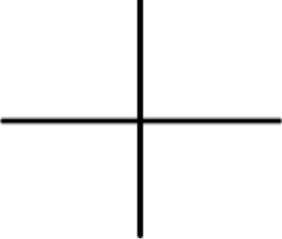
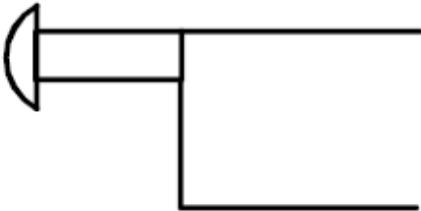
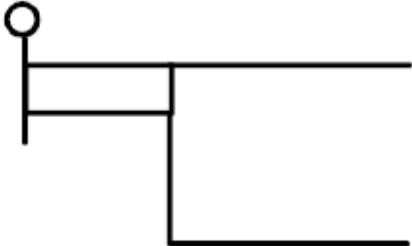
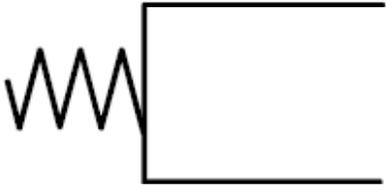
Simbologia (da UNI ISO 1219)

	Condotta di lavoro, condotta di pilotaggio dell'alimentazione, condotta di ritorno, condotta elettrica
	Condotta di pilotaggio (comando) interna e esterna. Condotta di drenaggio, spurgo o scarico. Filtro. Posizioni intermedie.
	Inquadramento di due o più funzioni riunite in un solo blocco o in una sola unità di montaggio.
	Connessioni meccaniche (albero, leva, stelo di pistone).
	Apparecchi di conversione dell'energia (pompa, compressore, motore).
	Strumenti di misura.
	Valvola di ritegno, giunto rotante, connessione meccanica, rotella (sempre con un punto al centro).
	Componenti di comando.

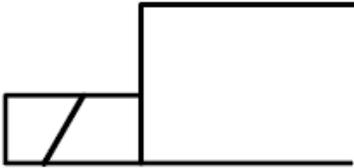
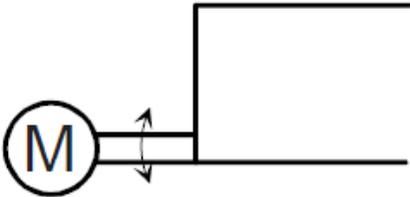
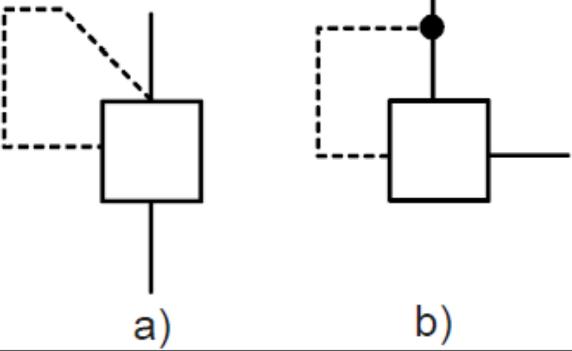
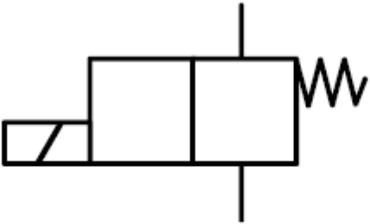
Simbologia (da UNI ISO 1219)

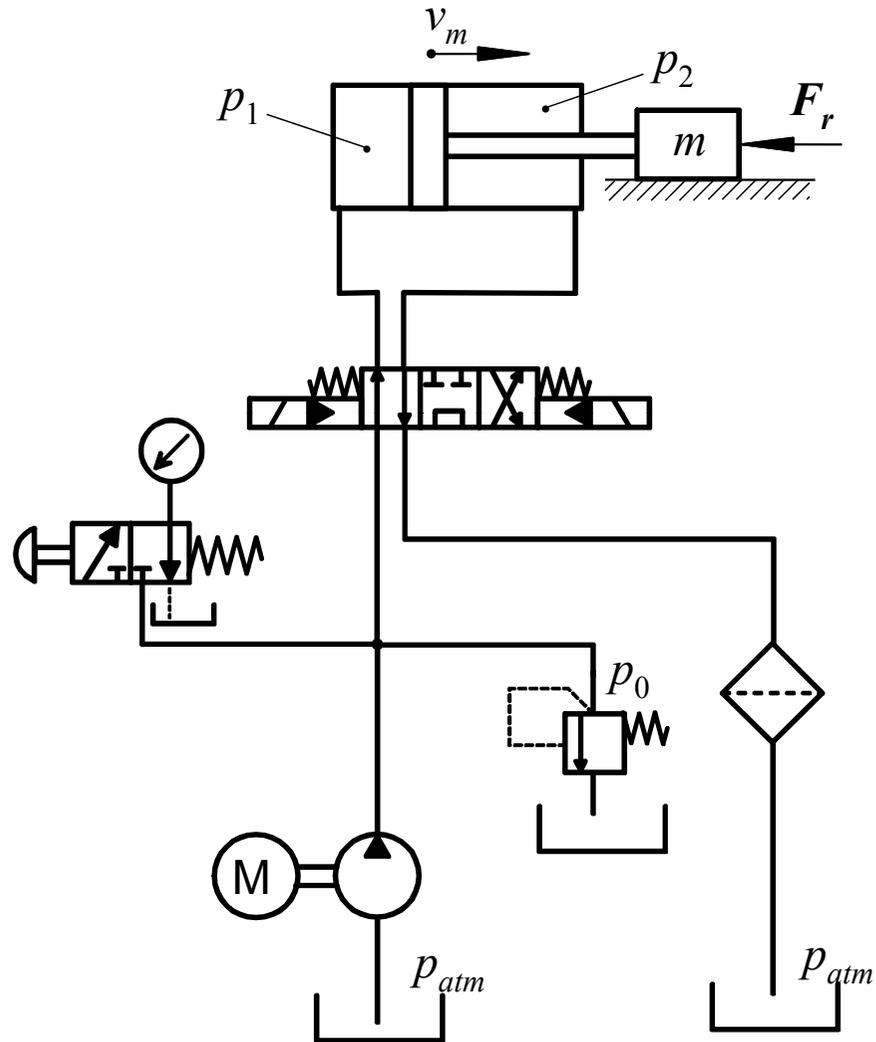
	Apparecchi di condizionamento (filtro, separatore, lubrificatore, scambiatore di calore).
	Serbatoio.
	Oleidraulico.
	Pneumatico.
	Raccordo.

Simbologia (da UNI ISO 1219)

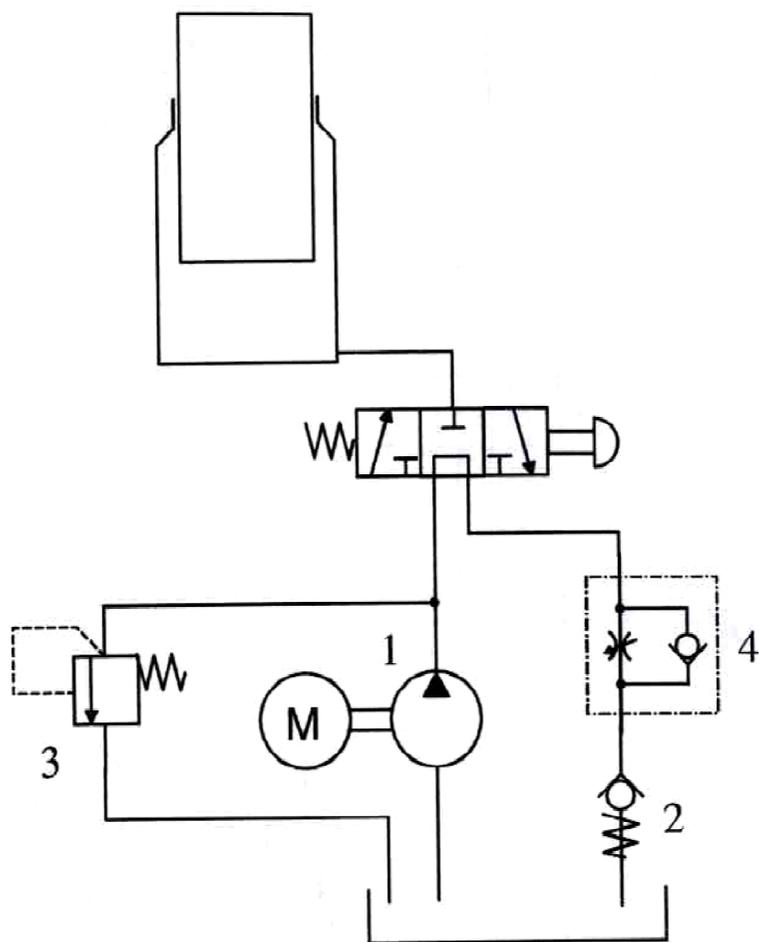
	Incrocio senza connessione.
	Comando manuale (pulsante).
	Comando manuale (leva).
	Comando meccanico (molla).

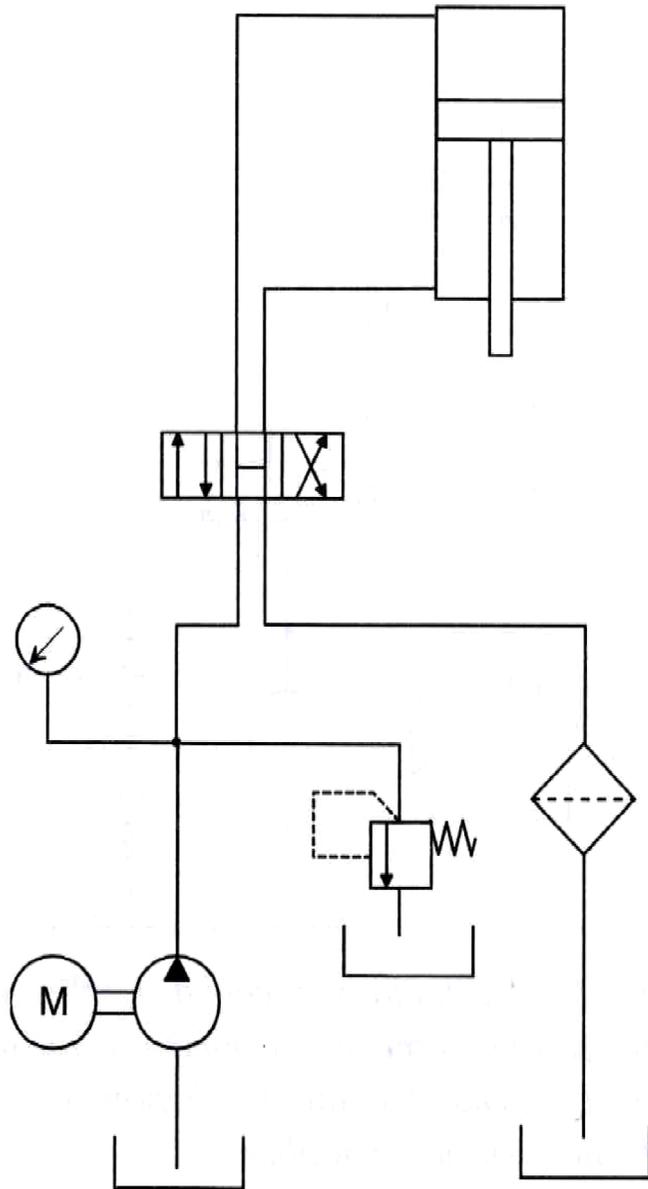
Simbologia (da UNI ISO 1219)

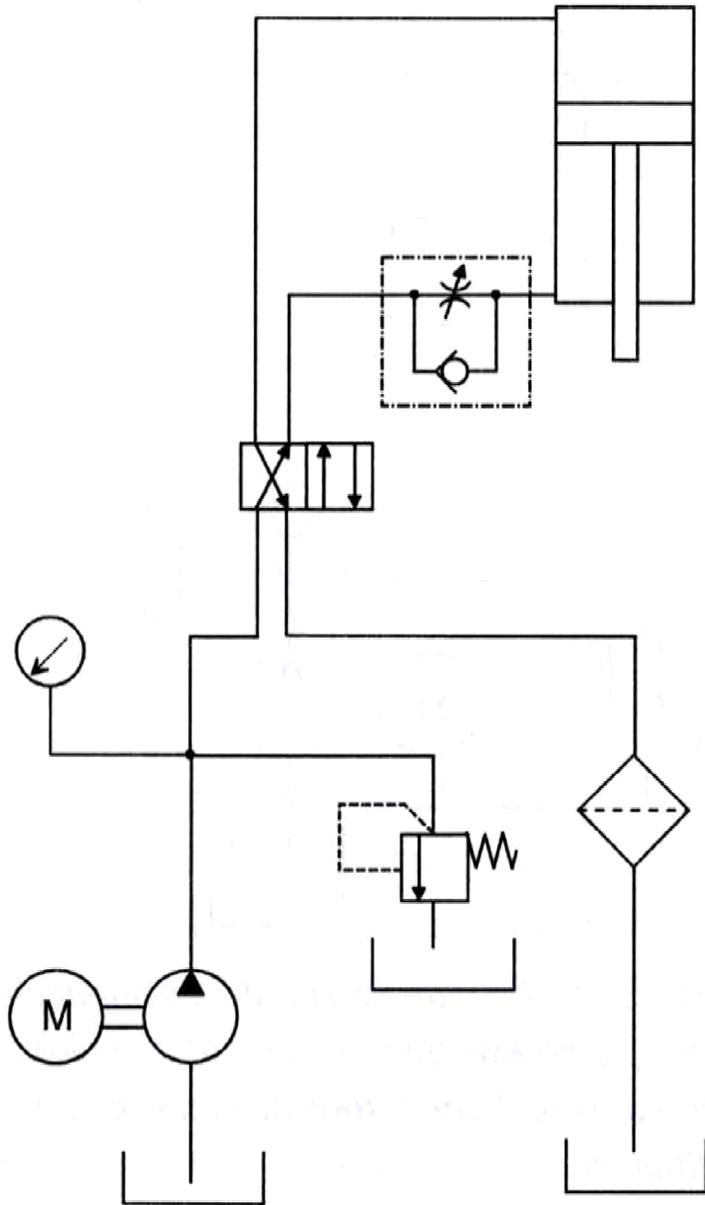
	<p>Comando elettrico (con 1 bobina).</p>
	<p>Motore elettrico.</p>
 <p>a) b)</p>	<p>Comando mediante applicazione o scarico della pressione. a) la via del comando è situata all'interno dell'unità; b) la via del comando è situata all'esterno dell'unità.</p>
	<p>Regole di applicazione dei simboli di comandi multipli nei simboli completi di apparecchiature. I simboli dei tipi di comando unidirezionale sono rappresentati adiacenti al simbolo dell'elemento comandato, in modo tale che le forze immaginarie agenti sul simbolo siano tali da muoverlo in un'altra posizione.</p>

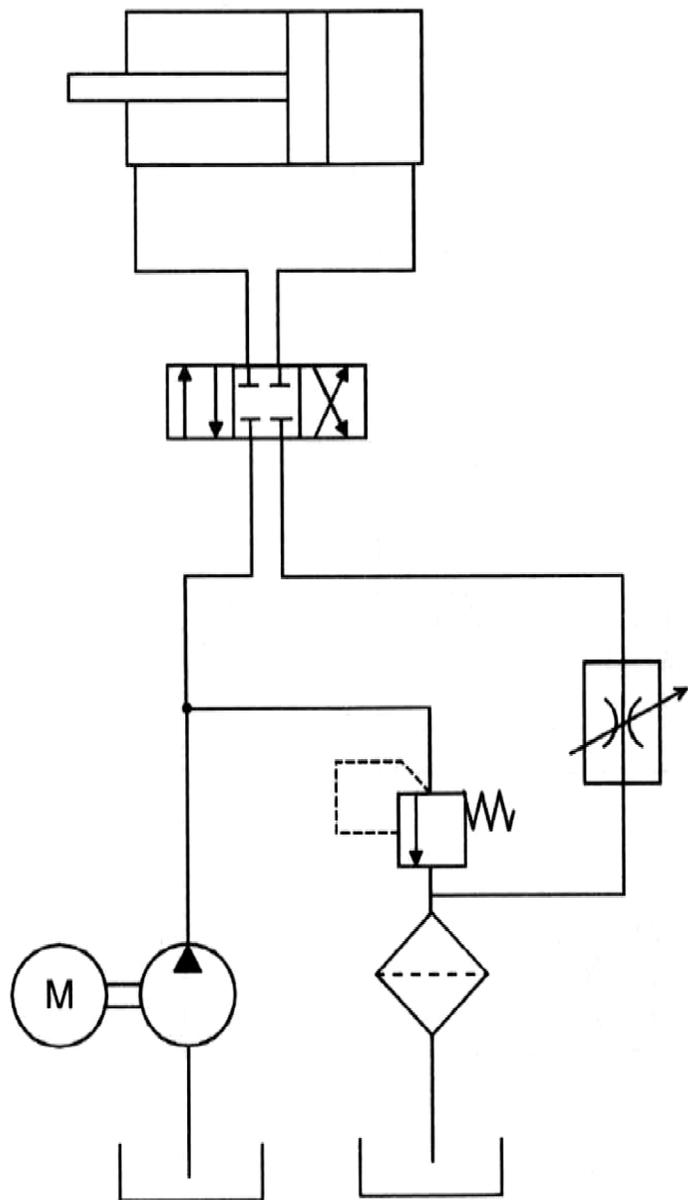


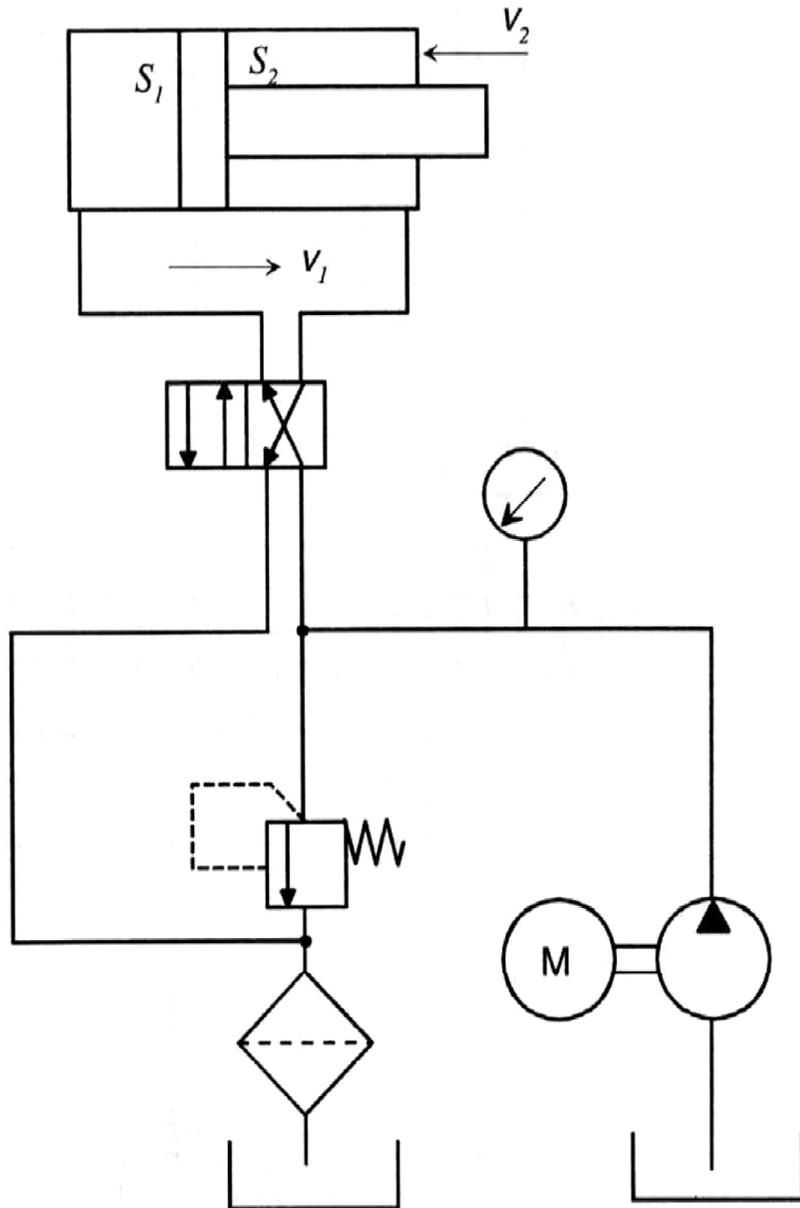
Circuiti elementari

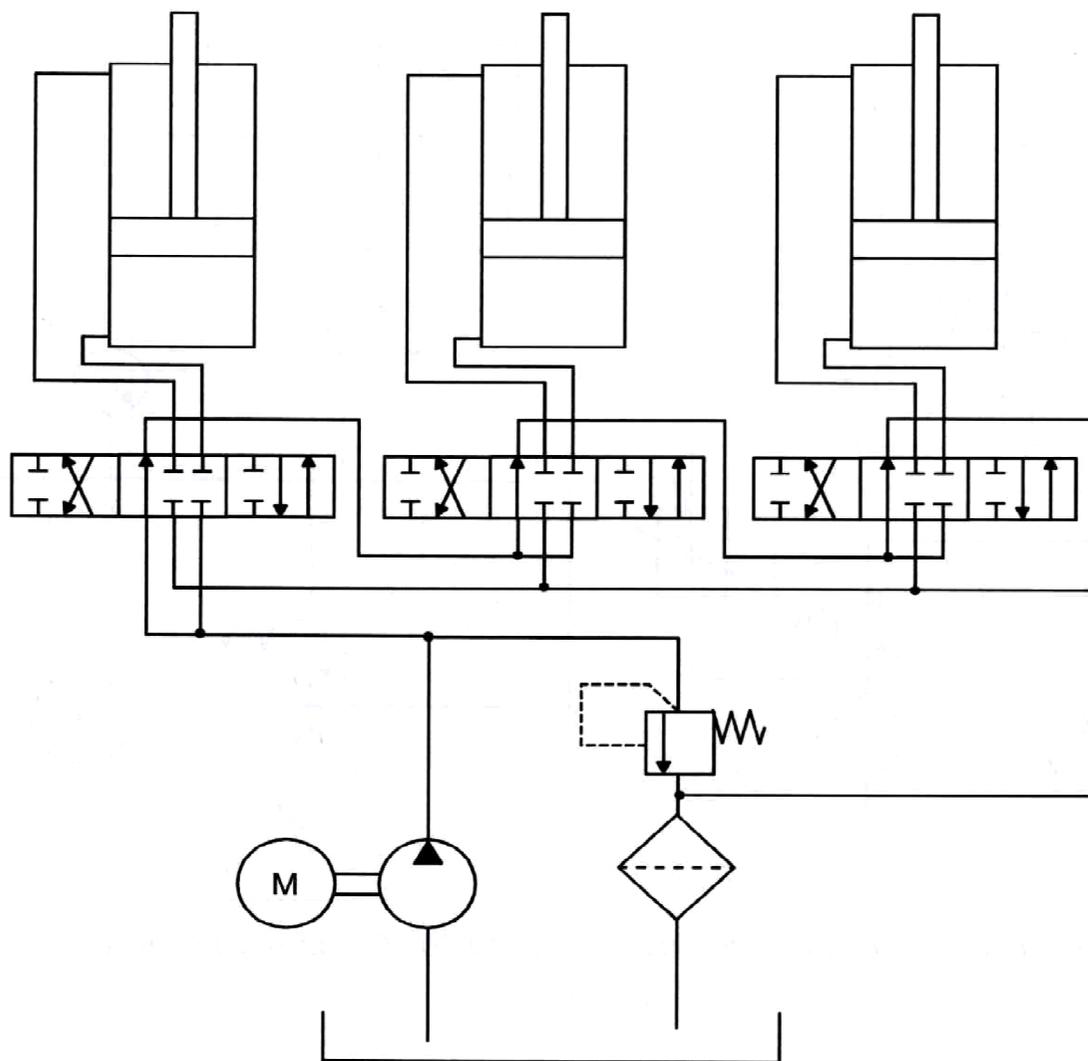


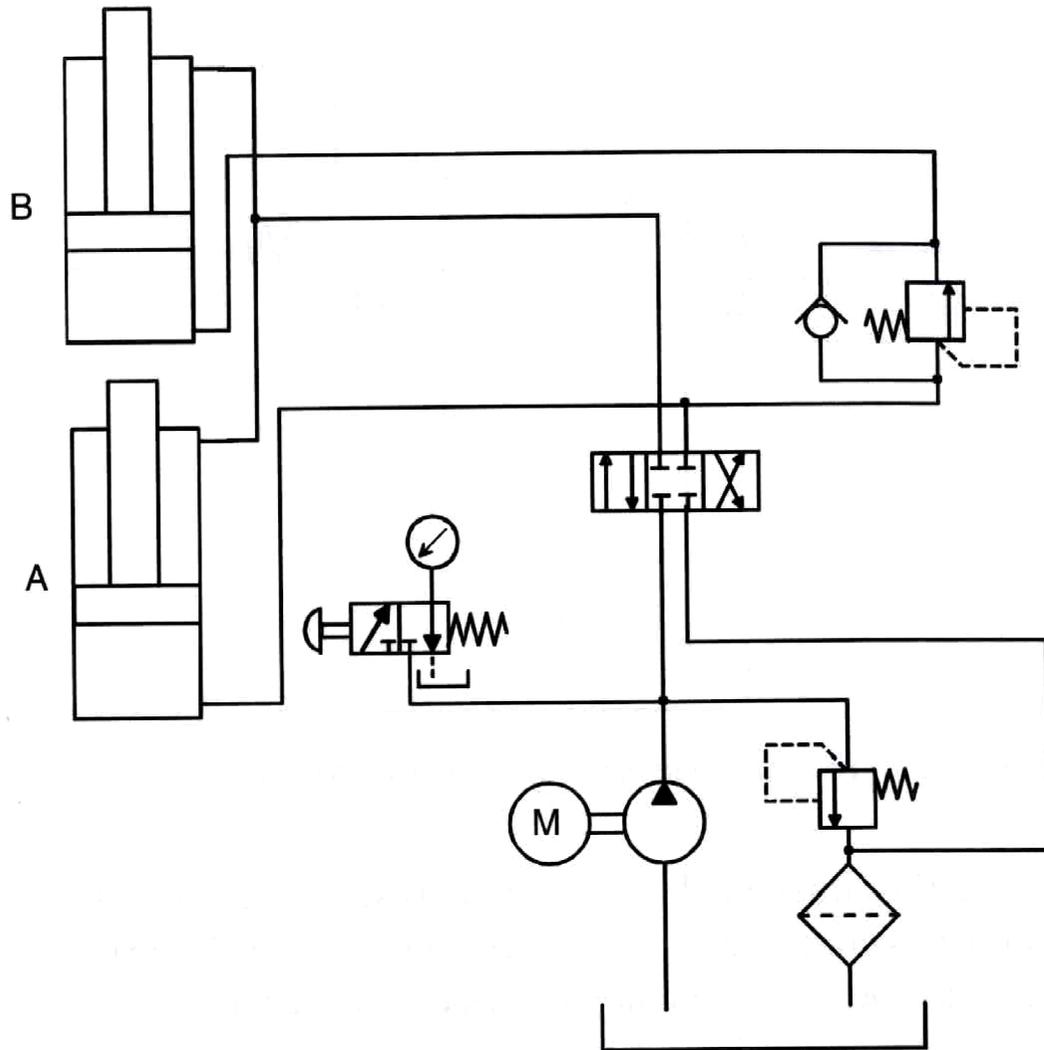




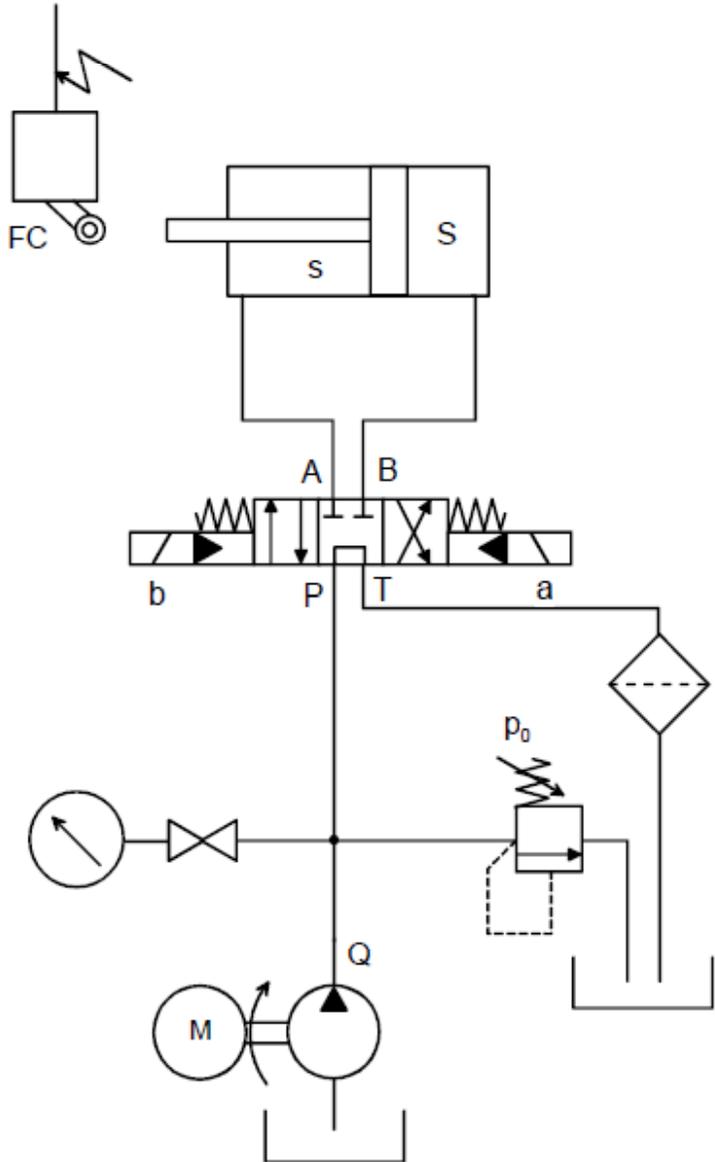




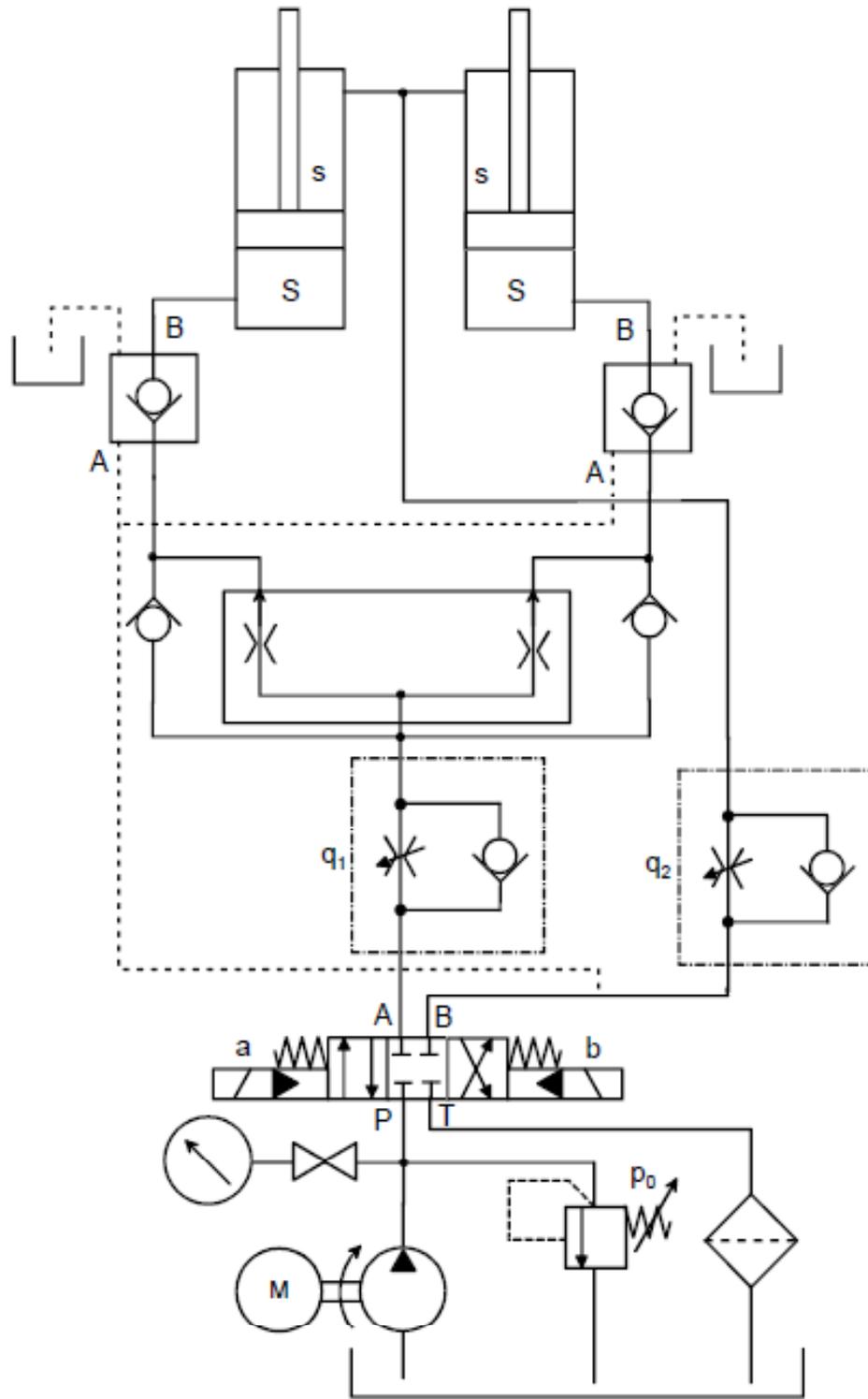




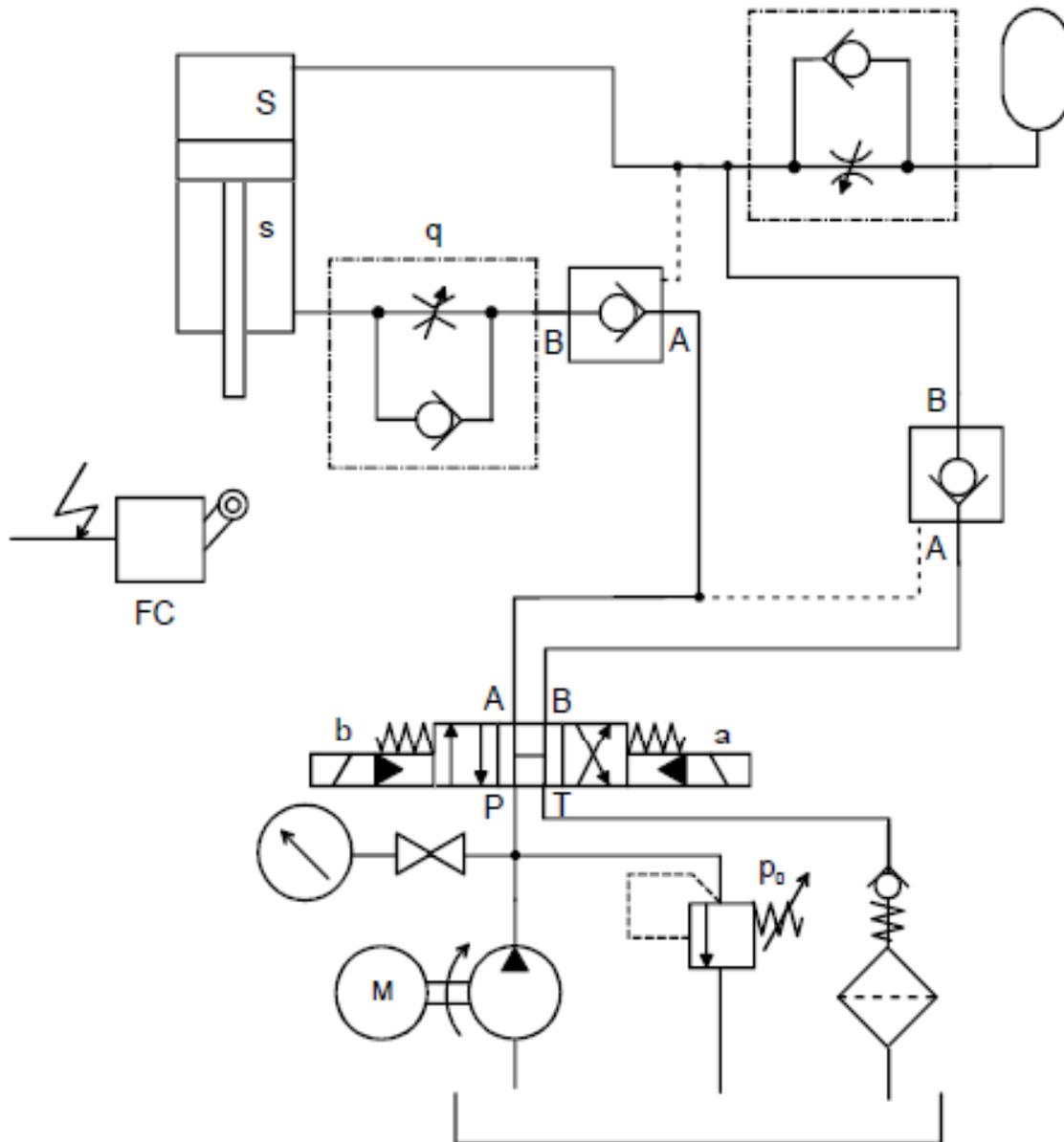
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Circuiti elementari



Circuiti elementari



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